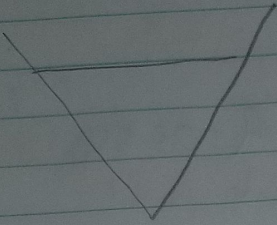


Test 3 Problem 3

Purpose - Design a triangular channel to be made of formed unfinished concrete to carry $5.75 \text{ m}^3/\text{s}$ of water when laid on a 1.2 percent slope

Drawings + DiagramsSources

Mott R, Untener, J.A. "Applied Fluid Mechanics" 7th edition
Pearson Education Inc. (2015)

Design Considerations

- Incompressible Fluid
- Steady State
- Isothermal problem

Data + Variables

$$Q = 5.75 \text{ m}^3/\text{s}$$

$$S = 1.2\% \text{ slope} = 0.012$$

$$n = 0.017$$

Test 3 Problem 3

Procedure & calculations

- 1) From the book, for designing an open channel's dimensions, you must use the following equation

$$\frac{nQ}{S^{1/2}} = AR^{2/3}$$

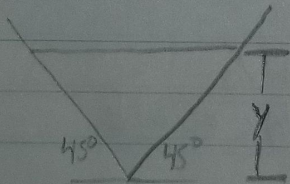
where $A = y^2$ and $R = 0.354y$ since we are designing a triangular channel

- 2) Plugging in known values into new equation

$$\frac{(0.017)(5.75 \text{ m}^3/\text{s})}{(0.012)^{1/2}} = (y^2)(0.354y)$$

$$0.8923 = (y^2)(0.354y)$$

$$y = 1.3609 \text{ m}$$



Summary the height of the triangle must be at least 1.3609m.

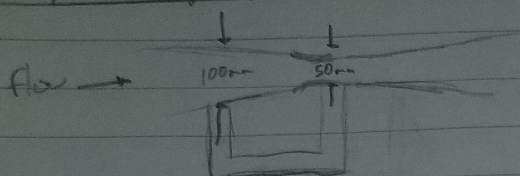
Material - concrete
- water

Analysis After working and modifying one of the examples out of the book, this problem was much easier to solve.

Test 3 Problem 5

Purpose - calculate volume flow rate in a certain venturi meter.

Drawings & Diagram



Sources

Mott R. Untener, J.A. "Applied Fluid Mechanics", 7th edition, Pearson Education Inc. (2015)

Design Considerations

- Incompressible fluid
- Steady State
- Isothermal problem

Data & Variables

$$\begin{aligned}
 D_1 &= 100 \text{ mm} = 0.1 \text{ m} \\
 D_2 &= 50 \text{ mm} = 0.05 \text{ m} \\
 T_w &= 80^\circ \text{C} \\
 \Delta p &= 55 \text{ kPa} \\
 \gamma_{\text{water @ } 80^\circ} &= 9.53 \text{ kN/m}^3 \\
 V &= 3.6 \times 10^{-7} \text{ m}^3/\text{s} \text{ (from table)} \\
 C &= 0.984
 \end{aligned}$$

Test 3 Problem 5

Procedure and calculations

- 1) Given the diameter at both sections, the area can be calculated

$$D_1 = 0.1 \text{ m}$$

$$D_2 = 0.05 \text{ m}$$

$$A_1 = \frac{\pi D_1^2}{4}$$

$$A_2 = \frac{\pi D_2^2}{4}$$

$$\underline{A_1 = 0.00785 \text{ m}^2}$$

$$\underline{A_2 = 0.00196 \text{ m}^2}$$

- 2) Velocity of flow calculation

$$V_1 = C \sqrt{\frac{2g(\Delta P)/\gamma}{(A_1/A_2)^2 - 1}}$$

$$= 0.984 \sqrt{\frac{(2(9.81 \text{ m/s}^2)(5000 \text{ Pa})/(9.53))}{(0.00785^2/0.00196^2) - 1}}$$

$$= 0.984 \sqrt{\frac{113.232}{15}}$$

$$\underline{V_1 = 2.70 \text{ m/s}}$$

- 3) checking C

$$N_R = \frac{V_1 D_1}{\nu}$$

$$= \frac{(2.70 \text{ m/s})(0.1 \text{ m})}{3.6 \times 10^{-7} \text{ s}^2/\text{m}}$$

$$N_R = 7.5 \times 10^5$$

$$N_R > 2 \times 10^5 \therefore C \text{ is correct}$$

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Test 3 Problem 5

4) calculating flow rate

$$Q = A_1 V_1$$

$$= (7.854 \times 10^{-3} \text{ m}^2) (2.70 \text{ m/s})$$

$$= 2.12 \times 10^{-2} \text{ m}^3/\text{s}$$

Summary

The volume flow rate under the circumstances is $2.12 \times 10^{-2} \text{ m}^3/\text{s}$

Materials

- Venturi meter
- water

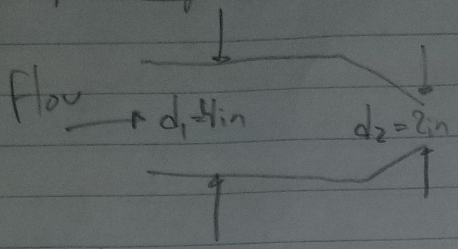
Analysis

The Volume flow rate makes sense for this problem as it is a relatively small venturi tube.

Test 3 Problem 6

Purpose - Calculate the force exerted by the water on the nozzle

Drawings & Diagrams



Sources

Mott, R. Untener, J.A. "Applied Fluid Mechanics" 7th ed.
Pearson Education Inc. (2015)

Design Considerations

- Incompressible fluid
- Steady State
- Isothermal problem

Data & Variables

$$d_1 = 4 \text{ in} = 0.33 \text{ ft}$$

$$d_2 = 2 \text{ in} = 0.16 \text{ ft}$$

$$K = 0.12$$

$$V_2 = 80 \text{ ft/s}$$

$$\rho = 1.94 \text{ lb/ft}^3$$

$$\gamma = 62.4 \text{ lb/ft}^3$$

$$g = 32.2 \text{ ft/s}^2$$

Test 3 Problem 6

Procedure + Calculations

- 1) First, the area of both the pipe and nozzle must be calculated.

$$A_1 = \frac{\pi d_1^2}{4}$$
$$= \frac{\pi \left(\frac{4}{12}\right)^2}{4}$$

$$A_1 = 0.0873 \text{ ft}^2$$

$$A_2 = \frac{\pi d_2^2}{4}$$
$$= \frac{\pi \left(\frac{2}{12}\right)^2}{4}$$

$$A_2 = 0.0218 \text{ ft}^2$$

- 2) Knowing the area of the nozzle and the velocity at its exit, the flow rate at the nozzle can be calculated.

$$Q = A_2 v_2$$
$$= (0.0218 \text{ ft}^2)(80 \text{ ft/s})$$

$$Q = 1.745 \text{ ft}^3/\text{s}$$

- 3) Since the fluid is incompressible, we can use the continuity equation to determine velocity in the tube.

$$A_1 v_1 = A_2 v_2$$

$$(0.0873 \text{ ft}^2)(v_1) = (0.0218 \text{ ft}^2)(80 \text{ ft/s})$$

$$v_1 = 19.977 \text{ ft/s}$$

Test 3 Problem 6

- 4) The equation for energy loss due to a sudden contraction is $h_L = K \times \frac{V^2}{2g}$

$$h_L = .12 \times \frac{80^2}{2 \times 32.2}$$

$$= .12 \times \left(\frac{6400}{64.4} \right)$$

$$= .12 \times 99.38$$

$$h_L = 11.925 \text{ ft}$$

- 5) Using the equation $\frac{P_A}{\gamma} + Z_A + \frac{V_A^2}{2g} - h_L = \frac{P_B}{\gamma} + Z_B + \frac{V_B^2}{2g}$ pressure in the hose can be calculated.

Rearranging the equation, we get

$$P_A = P_B + \gamma \left(\cancel{Z_A - Z_B} + \frac{(V_B^2 - V_A^2)}{2g} + h_L \right)$$

Z_A and Z_B are irrelevant to the equation

$$P_1 = \gamma \left(\frac{V_B^2 - V_A^2}{2g} + h_L \right)$$

$$= 62.4 \left(\frac{(80 \text{ ft/s})^2 - (19.977 \text{ ft/s})^2}{2 \times 32.2 \text{ ft/s}^2} + 11.925 \text{ ft} \right)$$

$$P_1 = 6558.68 \text{ lb/ft}^2$$

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Test 3 Problem 6

6) The equation $F = \rho Q(v_2 - v_1) + p_1 A_1 - p_2 A_2$ can be used to calculate the force exerted by the water on the nozzle.

$$F = (1.94)(1.743 \text{ ft}^3/\text{s})(-80 \text{ ft/s}) - (-19.977 \text{ ft/s}) + (6558.6816 \text{ lb/ft}^2)(0.0873 \text{ ft}^2)$$

$$F = 369.3816$$

Summary

The force exerted on the hose by the water is 369.3816.

Materials

- Hose
- Nozzle
- Water

Analysis