

9) Sin type 1c Copper Tube. 700 gpm
temp 77°F 1000 gpm

manometer 0-in-8 in scale

$$\Delta h = \frac{Q_{air}}{A} \quad D_{nozzle} = 4.805 \text{ in} \quad A = 0.1259 \text{ ft}^2$$

$$700 \text{ gpm} = \frac{0.1536 \times 1700}{60} \text{ ft}^3/\text{s} = Q_{air} \quad V_{air} = \frac{0.1536 \times 1700}{0.1259 \times 2} \text{ ft}/\text{sec} \quad V_{air} = 12.37 \text{ ft}/\text{sec} \checkmark$$

$$N_a = \frac{V D}{\nu} \quad \text{kinematic viscosity} = 3.84 \times 10^{-4} \text{ ft}^2/\text{s} \quad S_w = 5816 \text{ lb}/\text{ft}^3 \quad 4.805 \text{ in} \rightarrow 0.400 \text{ ft}$$

$$N_a = \frac{12.37 \cdot 0.400}{3.84 \times 10^{-4}} \quad N_a = 12,854.16 \quad C_D = 0.955$$

$$1000 \text{ gpm} = \frac{0.1536 \times 1000}{60} \text{ ft}^3/\text{sec} = Q_{max} \quad A = 0.1259 \text{ ft}^2$$

$$V_{max} = \frac{0.1536 \times 1000}{0.1259} \quad V_{max} = 17.68 \text{ ft}/\text{sec}$$

$$N_a = \frac{V D}{\nu} \quad \text{kinematic viscosity} = 3.84 \times 10^{-4} \text{ ft}^2/\text{sec} \quad 4.805 \text{ in} \rightarrow 0.400 \text{ ft}$$

$$N_a = \frac{17.68 \cdot 0.400}{3.84 \times 10^{-4}} \quad N_a = 18,416.4 \quad C_D = 0.961$$

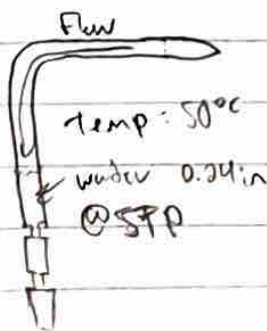
$$\gamma_m = 844.9 \text{ lb}/\text{ft}^3 \quad \gamma_w = 5816 \text{ lb}/\text{ft}^3 \quad h = 8 \text{ in} \rightarrow 0.667 \text{ ft} \quad g = 32.2 \text{ ft}/\text{s}^2$$

$$A_1 = \frac{A}{\sqrt{2gh(\gamma_w/\gamma_m)(C_D)^2} + 1} = \frac{0.1259}{\sqrt{2(32.2)(0.667)(844.9/5816)(0.961)^2} + 1} \quad A_1 = 0.07459 \text{ ft}^2$$

$$A_1 = \frac{\pi (d)^2}{4} \quad d = \sqrt{\frac{A_1 \cdot 4}{\pi}} \quad d = \sqrt{\frac{0.07459 \cdot 4}{\pi}} \quad d = 3.697 \text{ in}$$

15) $T = 50^\circ\text{C} = 122^\circ\text{F}$ manometer = 0.24 in (h) $\gamma = 1.184 \text{ lb}/\text{ft}^3$ $\gamma_w = 62.4 \text{ lb}/\text{ft}^3$
 $0.24 \text{ in} \rightarrow 0.02 \text{ ft}$

$$V_1 = \sqrt{2gh(\gamma_w/\gamma - 1)} \quad V_1 = \sqrt{2(32.2)(0.02)(62.4/1.184 - 1)} \quad V_1 = 8.16 \text{ ft}/\text{sec} \checkmark$$



HW. 2.3

Chapter 15

4) $D = 10$ in flow rate (Q) = 25 gal/min $SG = 0.83$ Dynamic Viscosity (μ) = 2.5×10^{-6} lb-s/in²
 Orifice diameter (d_1) = 1.0 in (d_2) = 7.0 in Specific Weight (γ_s) = 59.41 lb/ft³

$$a) \quad D_1 = \frac{\pi (1.94)^2}{4} \quad D_1 = 0.9418 \text{ ft}^2 \quad D_2 = \frac{\pi (0.937)^2}{4} \quad D_2 = 5.41 \times 10^{-3} \text{ ft}^2$$

$$\gamma_s = (0.83)(62.416/\text{ft}^3) \quad \gamma_s = 51.79 \text{ lb/ft}^3$$

Conversion from gal/min to ft³/s $\frac{25 \text{ gal/min}}{44.5 \text{ gal/min}} \cdot \frac{1 \text{ ft}^3/\text{sec}}{7.48 \text{ gal}} = 0.035 \text{ ft}^3/\text{sec}$

$$V_1 = \frac{Q}{A_1} = \frac{0.035}{0.941} \quad V_1 = 0.0375 \text{ ft/sec}$$

$$\rho_s = SG(\rho_w) = 0.83(1.94) = 1.6102 \text{ slugs/ft}^3$$

$$\text{Dynamic Viscosity} = \frac{2.5 \times 10^{-6}}{1.6102} = 1.55 \times 10^{-6} \text{ ft}^2/\text{s}$$

$$N_R = \frac{V_1 D}{\nu} = \frac{0.0375(0.83)}{1.55 \times 10^{-6}} = 5.14 \times 10^4$$

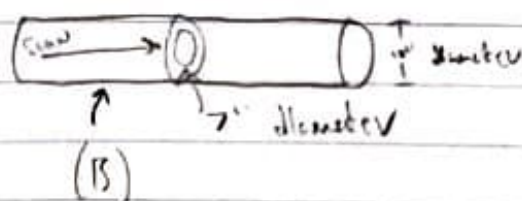
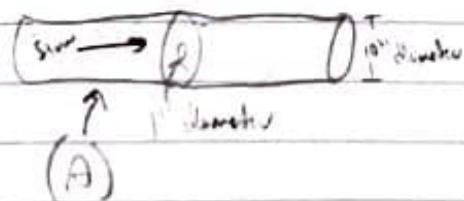
Reynolds number of (a) 1.0 in pipe $\beta = \frac{d}{D} = \frac{1.0}{10} \quad \beta = 0.1 \quad C_D = 0.594$

Reynolds

$$a) \quad h = \left(\frac{0.0375}{0.594} \right)^2 \frac{[(0.594/5.14 \times 10^{-4})^2 - 1]}{2(32.2)[(62.4/51.79) - 1]} \quad h = 23 \text{ ft} \checkmark$$

$$b) \quad A = \frac{\pi (0.58)^2}{4} \quad A = 0.264 \text{ ft}^2 \quad \beta = \frac{7.0}{10} \quad \beta = 0.7 \quad C_D = 0.618$$

$$h = \left(\frac{0.0375}{0.618} \right)^2 \frac{[(0.594/5.14 \times 10^{-4})^2 - 1]}{2(32.2)[(62.4/51.79) - 1]} \quad h = 0.00678 \text{ ft} \checkmark$$

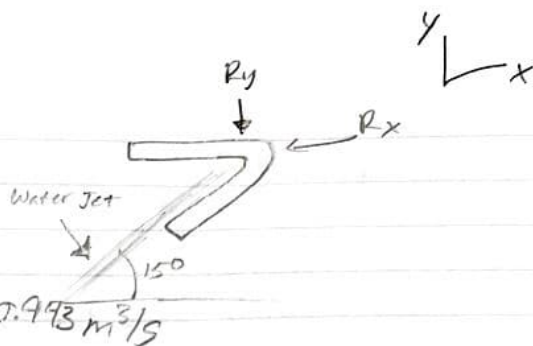


HW 2.3

Mason Firm

Chapter 16

16.20



$$Q = A_v = (3.142 \times 10^{-2}) (30 \text{ m/s}) = 0.943 \text{ m}^3/\text{s}$$

$$\underline{x} \quad R_x = \rho Q (v_2 - v_1) = \rho Q (v_2 - (-v_1 \cos 15^\circ))$$

$$R_x = \rho Q v (1 + \cos 15^\circ) = 1.966 \rho Q v$$

$$R_x = (1.966) \left(\frac{1000 \text{ kg}}{\text{m}^3} \right) \left(\frac{0.943 \text{ m}^3}{\text{s}} \right) \left(\frac{30 \text{ m}}{\text{s}} \right)$$

$$R_x = 55.618 \times 10^3 \text{ kg} \cdot \text{m/s}^2 = 55.618 \text{ kN}$$

$$\underline{y} \quad R_y = \rho Q (v_2 - v_1) = \rho Q (0 - (-v_1 \sin 15^\circ)) = \rho Q v \sin 15^\circ$$

$$R_y = (1000) (0.943) (30) (0.259) = 7.327 \times 10^3 \text{ kg} \cdot \text{m/s}^2 = 7.327 \text{ kN}$$

$$v_{1x} = v_1 \cos 15^\circ = (30 \text{ m/s}) (0.966) = 28.98 \text{ m/s}$$

$$v_{1y} = v_1 \sin 15^\circ = (30 \text{ m/s}) (0.259) = 7.76 \text{ m/s}$$

$$v_{c/x} = v_{1x} - 12 \text{ m/s} = 16.98 \text{ m/s}$$

$$|v_{c/y}| = \sqrt{(16.98)^2 + (7.76)^2} = 18.67 \text{ m/s}$$

$$\dot{m} = \rho Q = \rho A v_c = (1000 \text{ kg/m}^3) (3.142 \times 10^{-2} \text{ m}^2) (18.67 \text{ m/s}) = 586.6 \text{ kg/s}$$

$$\text{Velocity at angle} \rightarrow \alpha = \tan^{-1} (7.76 / 16.98) = 24.56^\circ$$

$$\beta = 24.56^\circ - 15^\circ = 9.56^\circ$$

$$V_{c1F}(V_{c1})\cos(9.56) = (14.67)(0.986) = 18.41 \text{ m/s}$$

$$R_x = M_c(\Delta V_x) = M_c(V_{c2x} - V_{c1x}) = (586.6)(18.41 - (-16.98))$$

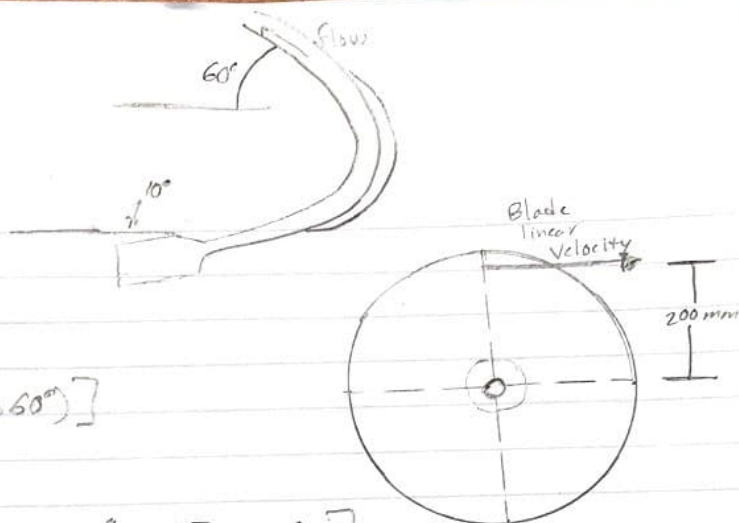
$$R_x = 20.86 \text{ kN}$$

$$R_y = M_c(\Delta V_y) = M_c(V_{c2y} - V_{c1y}) = (586.6)(0 - (-7.76))$$

$$R_y = 4.55 \text{ kN}$$

16.29

Given:
 $D = 7.50 \text{ mm}$
 $V = 25 \text{ m/s}$
 $H_2O \text{ Temp} = 15^\circ \text{C}$



$$f_x = \rho A V [V \cos 10^\circ - (-V \cos 60^\circ)]$$

$$10^3 \times \frac{\pi}{4} \times (7.5 \times 10^{-3})^2 \times 25 [25 \cos 10^\circ + 25 \cos 60^\circ]$$

$$= 410.997 \approx \boxed{41 \text{ N}}$$

$$f_y = \rho A V [V \sin 10^\circ - V \sin 60^\circ]$$

$$10^3 \times \frac{\pi}{4} \times (7.5 \times 10^{-3})^2 \times 25 \times [25 \sin 10^\circ - 25 \sin 60^\circ]$$

$$\boxed{-19.11 \text{ N}}$$

This is negative because the force in the vertical direction is exerted down.

HW 2.3

CH 16

16, 11, 20, 24

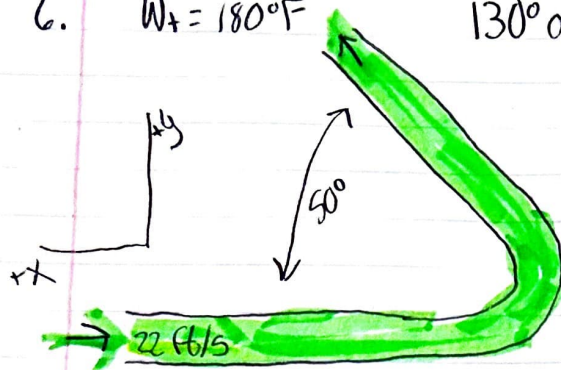
6. $W_t = 180^\circ F$

130° angle

$V = 22 \text{ ft/s}$

$\gamma_u = 62.4 \text{ lb/ft}^3$

$A = 2.95 \text{ in}^2$



Complete forces in x and y

$$Q = V \cdot A \therefore (22 \text{ ft/s}) \cdot 2.95 \text{ in}^2 = 778.8 \text{ in}^3/\text{s}$$

$$\dot{m} = \rho \cdot Q = 62.4 \text{ lb/ft}^3 \cdot 778.8 \text{ in}^3/\text{s}$$

$$.0361 \cdot 778.8 = 28.11 \text{ lb/s}$$

$$F_x = \dot{m} (22 \cos 50 - 22)$$

$$28.11 (22 \cos 50 - 22)$$

$$1015.43 \frac{\text{lb} \cdot \text{ft}}{\text{s}^2} \cdot .03108 = \boxed{31.58 \text{ lbf} = F_x}$$

$$F_y = \dot{m} (22 \sin 50 - 0) = 473.74 \cdot 12 = 5684.88 \frac{\text{lb} \cdot \text{ft}}{\text{s}^2}$$

$$\therefore 5684.88 \cdot .083$$

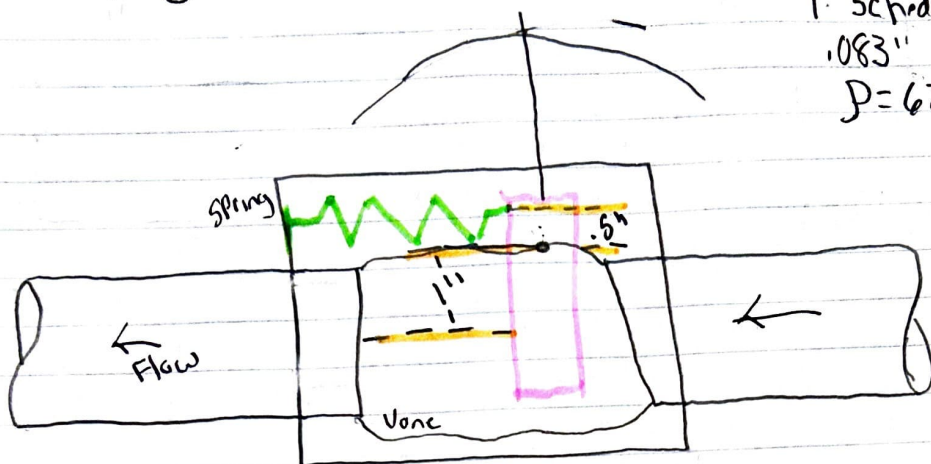
$$= \frac{471.845}{32.174} = \boxed{14.67 \text{ lbf} = F_y}$$

11. Calc Spring force needed to hold the Valve @ 100 gpm or .223 ft³/s

1" Schedule 40 Pipe

.083"

P = 62.4 lb/ft³



$$F_w = P \cdot A \cdot V^2$$

$$= P(Q) \cdot v$$

$$F_w = (62.4)(.223)(40.89)$$

$$F_w = 568.99 \text{ lb/ft}$$

$$V = \frac{.223}{\frac{\pi}{4} \left(\frac{1}{12} \right)^2} = 40.89 \text{ ft/s}$$

$$F_w \cdot l - S_F \cdot s = 0$$

$$S_F = 2 \cdot 568.99 = S_F = 1137.98 \text{ lb}$$