

4) Design Considerations

–waste heat rejected to the outside air

–The refrigerant leaves the evaporator superheat by 2.7°C .

–Sub-cooled by 6.3°C at the exit of the condenser.

–Isentropic efficiency at the compressor is 80%

5) Data and Variables

$$\dot{Q}_{\text{room}} = 72,000 \text{ Btu/hr}$$

$$\dot{V} = 500 \text{ CFM}$$

$$T_{\text{ub}} = 75^{\circ}\text{F}$$

$$\eta = 80\%$$

$$\phi = 50\%$$

$$\text{evaporator temp} = 2.7^{\circ}\text{C (superheat)}$$

$$\text{condenser temp} = 6.3^{\circ}\text{C (subcooling)}$$

6) Procedure

A) The evaporator pressure should be set so that the refrigerant boils at a temperature $10\text{--}15^{\circ}\text{F}$ below the indoor air temperature, ensuring effective heat absorption. Similarly, the condenser pressure should be selected so that the refrigerant condenses at a temperature $20\text{--}30^{\circ}\text{F}$ above the outdoor air temperature, allowing for efficient heat rejection to the environment.