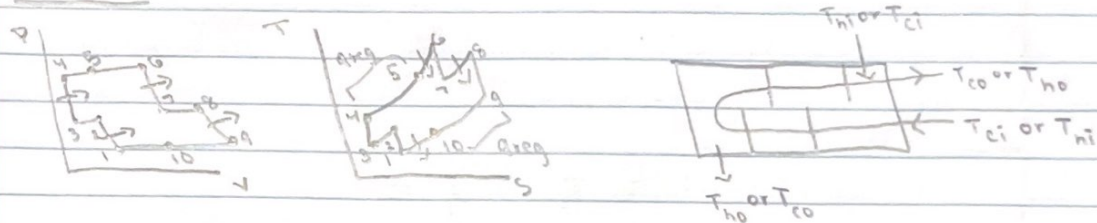


TEST 1 - MET 350 - KIANLA DAVIES

- ① PURPOSE: CALCULATE THE EFFECTIVENESS OF THE REGEN. HEAT EXCHANGER & COMPARE TO TYPICAL VALUES, FIND THE PRODUCED POWER IF TURNED OFF & DETERMINE WHY THE THERMAL EFFICIENCY DID/DID NOT CHANGE.

DIAGRAMS:



SOURCES:

- CENGEL & BOLES. THERMODYNAMICS - AN ENGINEERING APPROACH. 8TH EDITION MCGRAW HILL. 2015
- CANVAS NOTES/SLIDES

DESIGN CONSIDERATIONS

1) ASSUME THE FOLLOWING:

- 1) AIR BEHAVES AS AN IDEAL GAS; $R=0.3704$
- 2) REGENERATOR OPERATES PERFECTLY
- 3) ISENTROPIC COMPRESSION & EXPANSION
- 4) CONSTANT C_p , C_v AT ROOM TEMP
 $C_p=1.004976$ $C_v=0.717226$ $K=1.4$

DATA & VARIABLES

$T=17^\circ\text{C}$ $P=100\text{ kPa}$ @ Inlet, $r_p=4$, 300 kJ/kg heat added in each chamber,
 $W_{\text{net}}=226.801\text{ kJ/kg}$, $q_{\text{in}}=600\text{ kJ/kg}$, $\eta_{\text{TH}}=0.378$

MATERIALS

AIR AS AN IDEAL GAS

PROCEDURE:

I obtained all states first, and then calculated the net power produced by the cycle to find the thermal efficiency. The effectiveness was then calculated using the states found & compared to the values in Fig 11.12. POWER WAS FOUND TO NOT EXIST (Error)

CALCULATIONS:

From prob. 9-133: $\frac{P_2}{P_1} = \frac{P_4}{P_3} \neq \frac{P_6}{P_7} = \frac{P_8}{P_9}$

① isen	② CP	③ isen	④ CP	⑤ CP	⑥ isen	⑦ CP	⑧ isen	⑨ CP	⑩ CP
$P_1 = 100 \text{ kPa}$	$\frac{P_2}{P_1} = 4$	$P_3 = 400 \text{ kPa}$	$\frac{P_4}{P_3} = 4$	$P_5 = 1600 \text{ kPa}$	$P_6 = 1600 \text{ kPa}$	$\frac{T_6}{T_7} = \frac{P_6}{P_7}^{\frac{\gamma-1}{\gamma}}$	$P_8 = 400 \text{ kPa}$	$P_9 = 100 \text{ kPa}$	$P_{10} = 100 \text{ kPa}$
$T_1 = 290 \text{ K}$	$P_2 = 400 \text{ kPa}$	$T_3 = 290 \text{ K}$	$P_4 = 1600 \text{ kPa}$	$T_5 = 430.9 \text{ K}$	$T_6 = 430.9 \text{ K}$	$T_7 = \frac{1}{4} \times \frac{430.9}{1.4} = 269.97 \text{ K}$	$T_8 = 400 \text{ K}$	$T_9 = 430.9 \text{ K}$	$T_{10} = 100 \text{ K}$
	$\frac{T_2}{T_1} = \left(\frac{P_2}{P_1}\right)^{\frac{\gamma-1}{\gamma}}$		$\frac{T_4}{T_3} = \left(\frac{P_4}{P_3}\right)^{\frac{\gamma-1}{\gamma}}$						
	$T_2 = 290 \times 4^{\frac{0.4}{1.4}} = 430.9 \text{ K}$		$T_4 = 290 \times 4^{\frac{0.4}{1.4}} = 430.9 \text{ K}$						

$$W_{net} = W_{out,6-7} + W_{out,8-9} - W_{in,1-2} - W_{in,3-4}$$

$$W_{out,6-7} = C_p(T_6 - T_7) = 1.005(430.9 - 269.97) = 141.9 \text{ kJ/kg}$$

$$W_{out,8-9} = C_p(T_8 - T_9) = 1.005(400 - 430.9) = -31.05 \text{ kJ/kg}$$

$$W_{in,1-2} = C_p(T_2 - T_1) = 1.005(430.9 - 290) = 141.6 \text{ kJ/kg}$$

$$W_{in,3-4} = C_p(T_4 - T_3) = 1.005(430.9 - 290) = 141.6 \text{ kJ/kg}$$

$$W_{net} = 141.9 + (-31.05) - 141.6 - 141.6 = -103.1 \text{ kJ/kg}$$

$$\eta_{TH} = \frac{103.1}{300} = 0.344$$

calculating the difference:

$$\frac{0.376 - 0.344}{0.376} = 0.089 \rightarrow \boxed{8.99\%}$$

$$E_{reg} = \frac{h_{in}}{h_{out}} = \frac{T_5 - T_4}{T_9 - T_4} = \frac{430.9 - 430.9}{430.9 - 430.9}$$

$$E_{reg} = 0?$$

$$W_{max} = \frac{P}{\epsilon} = \frac{800}{0} = \text{DNE}$$

SUMMARY

comparison table

9-133	TEST 1	
226.501 kJ/kg	103.1 kJ/kg	W_{net}
600 kJ/kg	275.74 kJ/kg	Q_{in}
0.379	0.344	η_{TH}

ANALYSIS

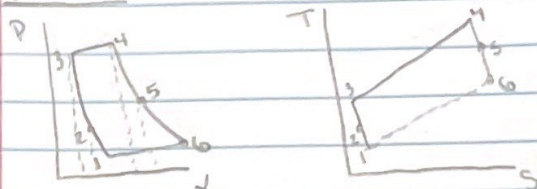
WHEN THE REHEATER WAS TURNED OFF, THE EFFICIENCY CHANGED 8.99%. REHEATING IS USED IN CYCLES TO INCREASE EFFICIENCY DUE TO THE DECREASE OF ENTROPY. THE EFFECTIVENESS WAS CALCULATED TO BE 0 WHICH MAKES SENSE DUE TO $C_{min}/C_{max} = 0$. HOWEVER, THAT ALLOWED FOR THE POWER TO NOT EXIST WHICH IS INCORRECT.

TEST 1 - MET 350 - KD

(2)

PURPOSE: CALCULATE THE PROPULSIVE EFFICIENCY OF THE JET PROPULSION CYCLE, THEN REPEAT USING DIFFERENT EFFICIENCIES, 1 VARIABLE CP/CV , + COMPARE TO THE PREVIOUS ONE.

DIAGRAMS:



SOURCES:

CENGEL & BOLES. THERMODYNAMICS - AN ENGINEERING APPROACH. 8TH EDITION. MCGRAW HILL 2015

CANVAS NOTES

DESIGN CONSIDERATIONS:

1. ASSUME THE FOLLOWING:

- 1) A PURE JET ENGINE
- 2) CP, CV CONSTANT @ ROOM TEMP
 $CP = 1.004926$, $CV = 0.717926$ $\gamma = 1.4$
- 3) IDEAL OPERATION OF ALL COMPONENTS

DATA & VARIABLES

$V_{inlet} = 240 \text{ m/s}$ $P_1 = 45 \text{ kPa}$ $T_1 = -13^\circ\text{C} = 260 \text{ K}$
 $D_{engine, inlet} = 1.6 \text{ m}$ $r_p = 13$ (compressor) $T_{inlet, turb} = 557^\circ\text{C} = 830 \text{ K}$
 $V_{engine, nozzle, exit} = 564.52 \text{ m/s}$ $F = 94392.21 \text{ N}$ $\eta_c = 80\%$ $\eta_t = 85\%$

MATERIALS

AIR AS AN IDEAL GAS

PROCEDURE

I STARTED BY FINDING ALL OF THE STATES (P,V,T) AND USED THE V-engine nozzle exit and F to CALCULATE THE PROPULSION EFFICIENCY IN PROB 9-142.

CALCULATIONS

$$P_3 = P_4 / P_1 = P_6 \quad \frac{P_2}{P_1} = 13 = \frac{P_2}{P_2}$$

①	1-2	②	2-3	③	3-4	④	4-5	⑤	5-6	⑥	6-7
$P_1 = 45 \text{ kPa}$		$T_2 = 260 + \frac{240^2}{2(0.717)}$		$T_3 = 600.99$		$P_4 = 845 \text{ kPa}$		$T_5 = T_4 - T_3 + T_2$		$P_6 = 45 \text{ kPa}$	
$T_1 = 260 \text{ K}$		$T_2 = 288.41 \text{ K}$		$P_3 = 13 \times 65$		$T_4 = 830 \text{ K}$		$T_5 = 517.81$		$\frac{T_6}{T_5} = \left(\frac{P_6}{P_5}\right)^{\frac{\gamma-1}{\gamma}}$	
$V_1 = 240 \text{ m/s}$		$P_2 = 45 \times \frac{288.41}{260} \times \frac{1}{1.4}$		$P_3 = 845 \text{ kPa}$		$V = 0.283 \text{ m}^3/\text{kg}$		$\frac{P_5}{P_4} = \left(\frac{T_5}{T_4}\right)^{\frac{\gamma}{\gamma-1}}$		$T_6 = 517.81 \left(\frac{45}{845}\right)^{\frac{1}{1.4}}$	
		$P_2 = 65 \text{ kPa}$						$P_5 = 167.1 \text{ kPa}$		$T_6 = 359 \text{ K}$	
		$V_2 = 0 \text{ m/s}$									

$$P_{\text{ideal}} = F + V_{\text{eng}} = 94392.2 \times 564.52 = 53286284.74 \text{ Nm/s} \rightarrow (1 \text{ Nm/s} = 1 \text{ W})$$

$$\eta_P = \frac{F}{P_{\text{ideal}}} = \frac{94392.2}{53286284.74} = 0.001 \text{ or } 0.17\%$$

DIDNT HAVE TIME TO FINISH

SUMMARY

THE EFFICIENCY OF JET CYCLE WAS FOUND TO BE 0.17%.

ANALYSIS

ALTHOUGH I DIDNT FINISH, I CAN PREDICT THAT THE EFFICIENCY WOULD INCREASE DUE TO THE VELOCITY AT THE EXIT OF THE NOZZLE BEING MINIMAL.