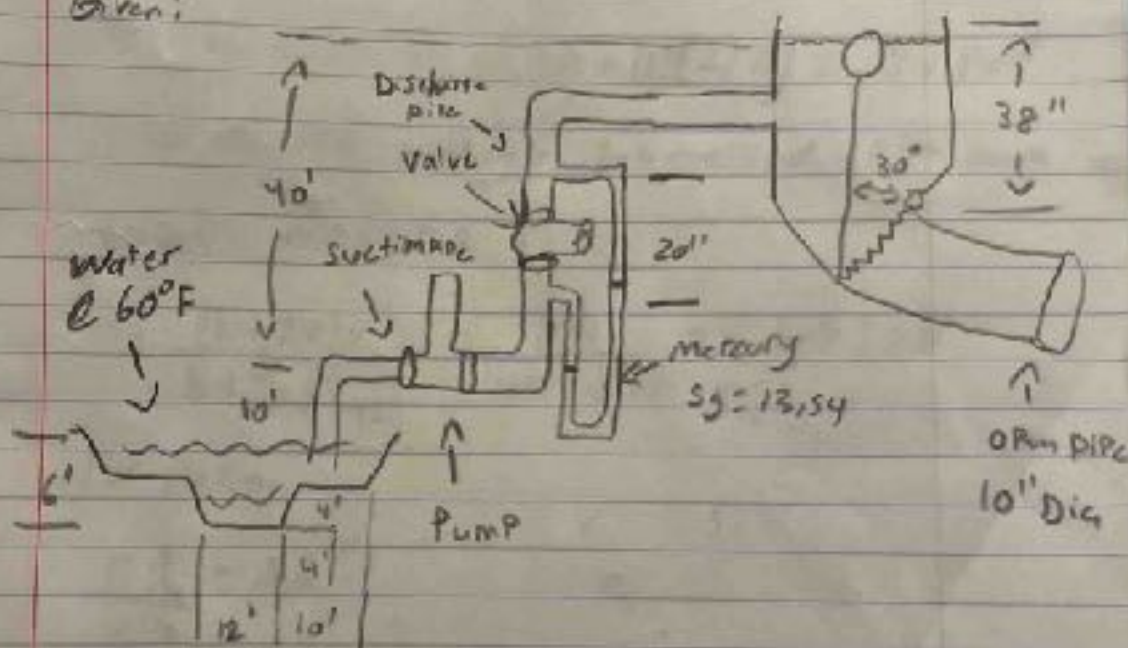


Test 1

Tyler Gray

Purpose: Design a system to deliver MET 330
60°F water from an open channel to
an elevated open channel at 3.38 ft³/s

Given:



Pump efficiency = 60%

Suction Pipe length = 11 ft

Discharge Pipe length = 2500 ft

Valve resistance = $K = 5.3$

Solution :

Area of ~~the~~ Par-channel

Trapezoid :

$$A = (b + zy)y = (40 + (1)(2)2) = 84 \text{ ft}^2$$

as the pipe was only submerged 2ft deep

hydraulic radius $= R =$

$$R = \frac{(b + zy)y}{b + zy\sqrt{1+z^2}} = \frac{84}{40 + 4\sqrt{1+1^2}} = 1.8398$$

For design

$$\frac{nQ}{s^{1/2}} = AR^{2/3}$$

it was not specified but n was taken
as clean excavated earth $n = 0.022$ from
table 14.1

Slope was not specified so typical slope
of 1% was used

Solving for Q

$$\frac{0.022 Q}{0.01^{1/2}} = (84)(1.8398)^{2/3}$$

$$\frac{0.022 Q}{0.01^{1/2}} = 126.123$$

$$0.022 Q = 12.612$$

$$Q = 573.28 \text{ Ft}^3/\text{min}$$

Max Flow rate given by open channel must not exceed

$$Q = 9.555 \text{ Ft}^3/\text{s}$$

this Q is used to solve for pipe diameter using Bernoulli's equation & $Q = VA$

We want a Fluid velocity of close to 3 m/s or 9.84 ft/s

Suction Pipe is 11 ft with 1 90° bend
Discharge Pipe is 2500 ft

$$h_A + \frac{P_1}{\gamma_1} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\gamma_2} + \frac{V_2^2}{2g} + Z_2 + h_{L12}$$

Determine: required Pump power in HP
Assume 60% p

$$h_A + \frac{P_1}{\gamma_1} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\gamma_2} + \frac{V_2^2}{2g} + Z_2 + h_{L12}$$

$$P_m = \frac{P_A}{\rho_A} = \frac{\gamma Q h_A}{\rho_A}$$

$$h_A = \frac{\cancel{P_2} - \cancel{P_1}}{\gamma} + \frac{V_2^2 - V_1^2}{2g} + Z_2 - Z_1 + h_{L12}$$

↑
open channel

$$Q = VA$$

$$8.5495 = 9.84 A$$

$$A = 0.344 \text{ ft}^2$$

K was Given as 5.3

h_L can be solved using the equation

$$h_{L12} = f \frac{L}{D} \frac{V^2}{2g} + \underset{\substack{\uparrow \\ \text{elbows } 90^\circ}}{3 \times K} \frac{V^2}{2g}$$

it was specified that a schedule 40 pipe was used because this was from table F.1

From table F.1

$$A = 0.344 \text{ ft}^2 \approx 0.317 \text{ ft}^2$$

a 18 in nominal Pipe size was selected
based from the chart

With an inside diameter of $0.6354 \text{ ft} = D$

$$Re = \frac{VD}{\nu} \quad 60^\circ\text{F water}$$

$\nu = 1.21 \times 10^{-5} \text{ from table A.2}$

$$Re = \frac{(9.84 \text{ ft/s})(0.6354 \text{ ft})}{1.21 \times 10^{-5}} = 516721.98 = 5.17 \times 10^5$$

$$\epsilon = \text{assumed Steel pipe} = 1.5 \times 10^{-4} \text{ ft}$$

$$\frac{D}{\epsilon} = \frac{0.6354}{1.5 \times 10^{-4}} = \underline{4236} \approx 10^4$$

From Moody's chart

$$f = 0.016$$

Plugging those found values into
the equation gives

$$h_L = 0.016 \cdot \frac{2500+11}{0.6354} \times \frac{9.84^2}{2(32.2)} + 3 \left(5.3 \times \frac{9.84^2}{2(32.2)} \right)$$

$$h_L = 63.23 \times 1.504 + 23.91$$

$$h_L = 118.98 \text{ ft}$$

$$h_A = Z_2 - Z_1 + 118.98$$

$$h_A = 50 \text{ ft} + 118.98 \text{ ft}$$

$$h_A = 168.98 \text{ ft}$$

Knowing Pump head I can solve for
hydraulic required power using

$$P_A = \gamma Q h_A$$

$$P_A = (62.4)(3387)(168.98)$$

$$P_A = 35713 \frac{\text{lb} \cdot \text{ft}}{\text{s}} \times \frac{1 \text{ HP}}{550 \text{ lb} \cdot \text{ft}} = 64.9 \text{ HP}$$

assuming $\eta = 60\%$

$$P_{\text{Pump}} = \frac{64.9}{.6} = 108.2 \text{ HP} \quad (A)$$

$$\frac{\Delta p}{\gamma} = K \frac{V^2}{2g} = h_L$$

$$\frac{P_1}{\gamma} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\gamma} + \frac{V_2^2}{2g} + Z_2 + h_L$$

$$\Delta p = \gamma h \text{ for mercury}$$

$$\frac{Q^2}{2g} \left(\frac{1}{A^2} - \frac{1}{A_B^2} \right) = \frac{P_B - P_A}{\gamma} + Z_A - Z_B \quad \begin{matrix} \text{same } \downarrow \\ \text{20 in} \end{matrix}$$

$$\frac{P_B - P_A}{\gamma_{\text{water}}} = Z_A - Z_B + \left(1 - \frac{S_{gH_2O}}{S_{gW}} \right) h$$

$$Q^2 = 2g \left(1 - \frac{S_{gH_2O}}{S_{gW}} \right) h + Z_A - Z_B$$

$$h = \frac{Q^2 - (Z_A - Z_B)}{1 - \frac{S_{gH_2O}}{S_{gW}}}$$

$$h = \frac{3.387^2 - 1.66 \text{ ft}}{1 - \frac{1}{13.54}} = 10.58 \text{ in}$$

$$h = \frac{3.387^2 - 1.666}{1 - \frac{1}{13.52}} = 10.58 \text{ in}$$

$$\frac{10.58 \text{ in}}{20 \text{ in}} = 0.529 = 52.9\%$$

(B)

Sources: "Applied Fluid Mechanics"
Robert L. Mott / Joseph A. Untener

Material list:

Steel Pipe Schedule 40 8in nominal size

Pump

U-Tube manometer

mercury

Water @ 60°F

Analysis:

This design is assuming a Steady Flow open channel and Pump efficiency is calculated at 60%. Results could vary based on things changing such as

- increased/decreased water temp
- change in open flow channel
- Pump efficiency dropping over time

Summary:

Water was to be moved from a lower open channel to a elevated open channel. The flow rate was to be $3.387 \text{ ft}^3/\text{s}$ to be delivered. Alternative Pump selection and their Flow rates was calculated with excel and a graph was created to visualize the relationship of Q by request. This could be beneficial if in the future a different flow rate was asked for. The graph and calculations would give the new pumps required HP while still taking consideration of the rest of the design.