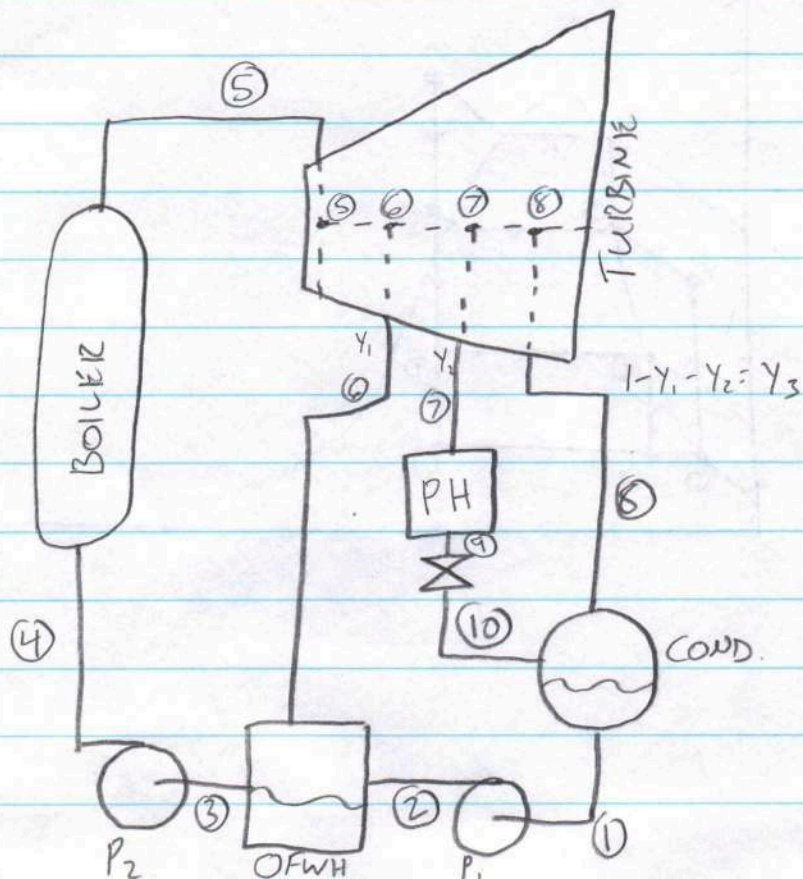


## GOLDMAN TEST 2

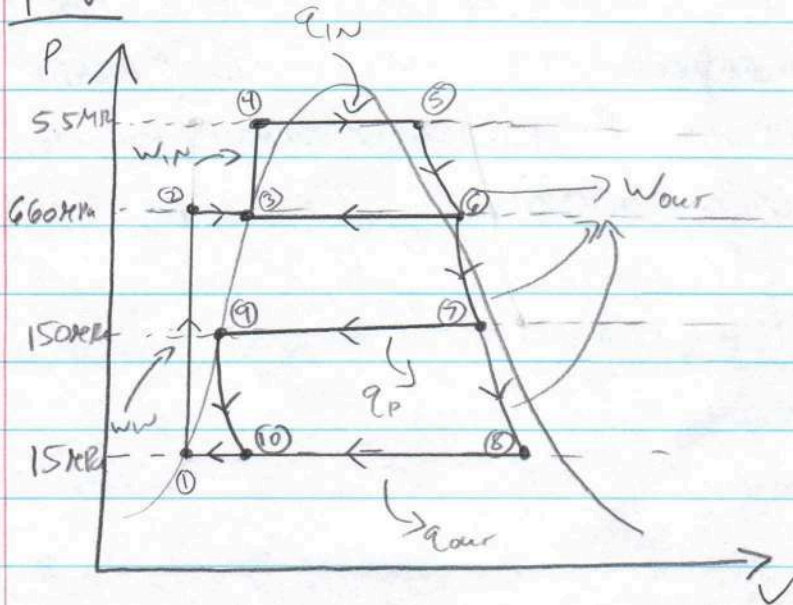
1. PURPOSE: THE PURPOSE OF THIS TEST/PROBLEM IS TO DEMONSTRATE THE ABILITY TO CREATE A SYSTEM FROM A DESCRIPTION, SOLVE FOR EACH STATE, AND OTHER VARIABLES, SPECIFICALLY:

- a)  $E_u$
- b) P-V, T-S DIAGRAMS
- c)  $W_{Turb}$
- d)  $\dot{m}_{str}$
- e)  $q_p$
- f) ALL STATES
- g)  $Y_2, Y_1$
- h)  $q_{out}$

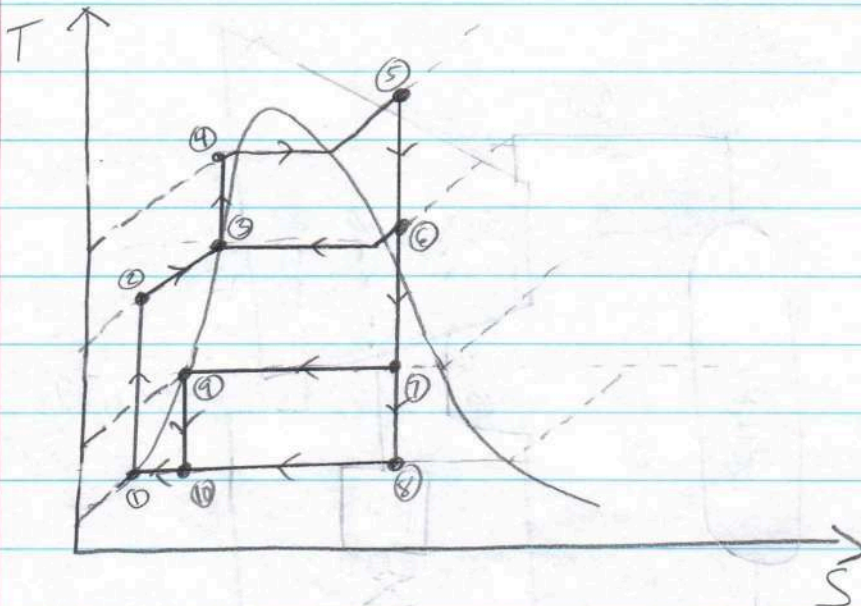
2. DRAWINGS:



P-V



T-S





- ④ SOURCES: "THERMODYNAMICS, AN ENGINEERING APPROACH" 6TH EDITION, YUNUS A. CENGAL  
MICHAEL A. BOLES

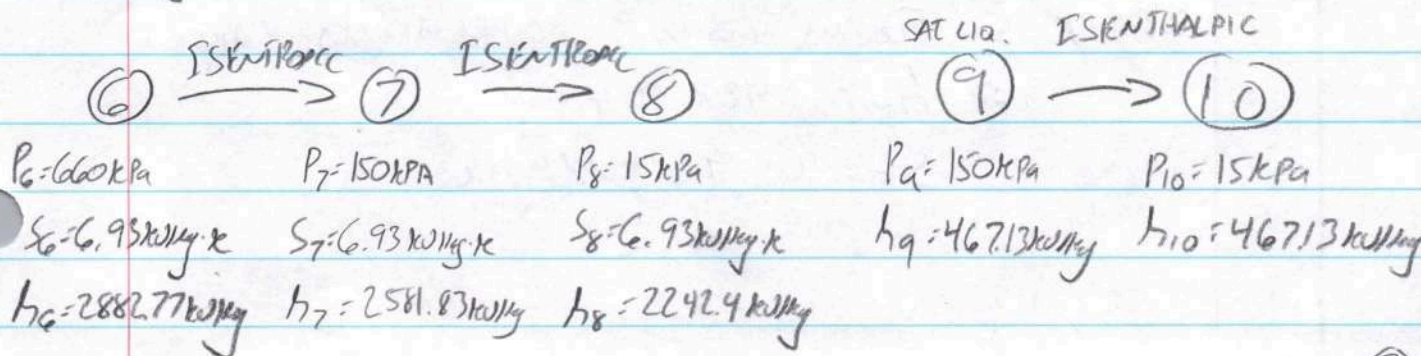
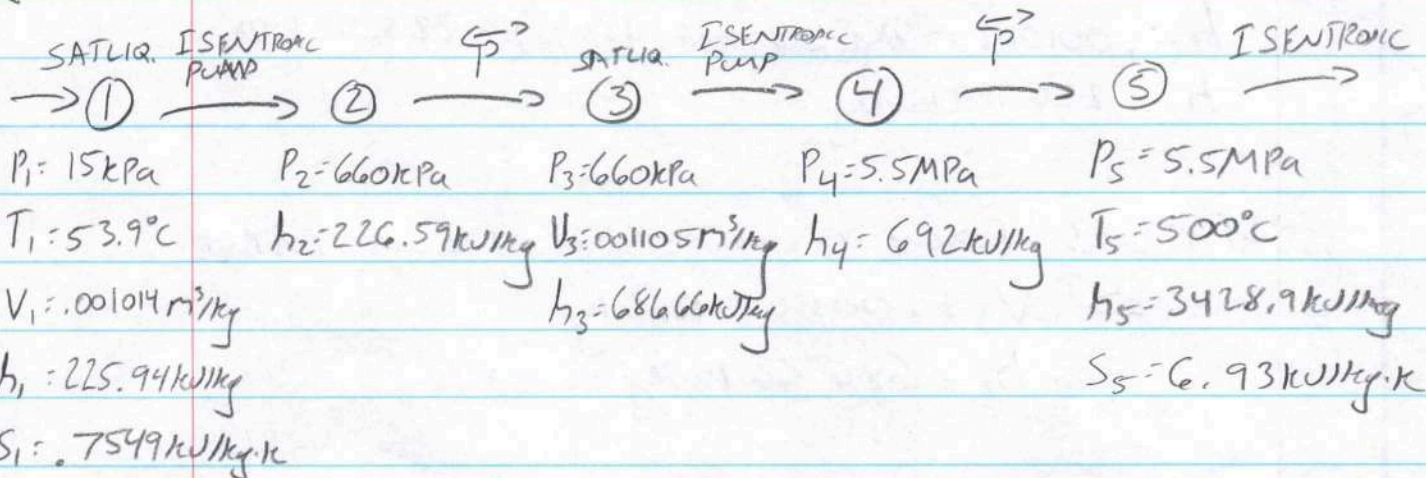
ODU CLASS NOTES.

④ DESIGN CONSIDERATIONS:

ASSUMPTIONS: - ALL LIQUID GOING INTO A PUMP IS SATURATED

- ALL STEAM ENTERING A TURBINE IS SUPERHEATED
- $y_1 = y_2$

(F) ⑤ DATA AND VARIABLES



6. PROCEDURE: - SOLVE STATES  
7. - ANSWER PROBLEM ASKED.

7. CALCULATIONS:

STATE 1 FOUND USING SAT LIQUID TABLE.

$$\Rightarrow T_1 = 53.9^\circ\text{C}$$

$$V_1 = 0.001014 \text{ m}^3/\text{kg}$$

$$h_1 = 225.94 \text{ kJ/kg}$$

$$s_1 = 0.7549 \text{ kJ/kg}\cdot\text{K}$$

FROM 1ST LAW:

$$h_2 = V_1(P_2 - P_1) + h_1$$

$$h_2 = 0.001014 \text{ m}^3/\text{kg}(660 \text{ kPa} - 15 \text{ kPa}) + 225.94 \text{ kJ/kg}$$

$$h_2 = 226.59 \text{ kJ/kg}$$

STATE 3 FOUND USING SAT LIQUID TABLE

$$\Rightarrow V_3 = 0.001105 \text{ m}^3/\text{kg}$$

$$h_3 = 686.66 \text{ kJ/kg}$$

STATE 5 FOUND USING SUPERHEATED TABLES.

$$\Rightarrow h_5 = 3428.9 \text{ kJ/kg}$$

$$s_5 = 6.93 \text{ kJ/kg}\cdot\text{K}$$



STATE 5  $\rightarrow$  6  $\rightarrow$  7  $\rightarrow$  8 ARE ISENTROPIC

$$\Rightarrow S_5 = S_6 = S_7 = S_8$$

I NEED TO FIND THE QUALITIES OF THESE STATES.

$$S_6 = S_F + X S_{Fg}$$

$$6.93 \text{ kJ/kg} \cdot \text{K} = 1.9682 \text{ kJ/kg} \cdot \text{K} + X(4.75898 \text{ kJ/kg} \cdot \text{K})$$

$$X_6 = 1.04 \Rightarrow \text{SUPERHEATED}$$

$$S_7 = S_F + X S_{Fg}$$

$$6.93 \text{ kJ/kg} \cdot \text{K} = 1.4337 \text{ kJ/kg} \cdot \text{K} + X(5.7894 \text{ kJ/kg} \cdot \text{K})$$

$$X_7 = .95 \Rightarrow \text{MIX}$$

$$S_8 = S_F + X S_{Fg}$$

$$6.93 \text{ kJ/kg} \cdot \text{K} = .7549 \text{ kJ/kg} \cdot \text{K} + X(7.2522 \text{ kJ/kg} \cdot \text{K})$$

$$X_8 = .85 \Rightarrow \text{MIX}$$

$h_6$  FOUND USING INTERPOLATION AND ONLINE CALCULATOR.

$$h_6 = 2882.77 \text{ kJ/kg}$$

$$h_7 = h_F + X h_{Fg}$$

$$h_7 = 467.13 \text{ kJ/kg} + (.95) 2226 \text{ kJ/kg}$$

$$h_7 = 2581.83 \text{ kJ/kg}$$

$$h_8 = h_F + X h_{Fg}$$

$$h_8 = 225.94 \text{ kJ/kg} + (.85) 2372.3 \text{ kJ/kg}$$

$$h_8 = 2242.4 \text{ kJ/kg}$$

STATE 9 FOWNS USING SAT LIQUID TABLES

$$h_9 = 467.13 \text{ kJ/kg}$$

STATE 10 IS AFTER A THROTTLE VALVE.

THIS DOES NOT AFFECT  $h$  (ISENTHALPIC)

$$\Rightarrow h_9 = h_{10}$$

$$h_{10} = 467.13 \text{ kJ/kg}$$

$$h_4 = v_3 (P_4 - P_3) + h_3$$

$$h_4 = .001105 \text{ m}^3/\text{kg} (5500 - 660) \text{ kPa} + 686.66 \text{ kJ/kg}$$

$$h_4 = 692 \text{ kJ/kg}$$

NOW WE NEED TO FIND  $y_1, y_2$ .

$$\text{WE KNOW } y_1 = y_2$$

$$\text{AND } 1 - y_1 - y_2 = y_3$$

$$\text{AND } y_1 + y_2 + y_3 = 1$$

WE CAN USE 1ST LAW TO GATHER EQUATIONS



$$\dot{m}h_3 = \dot{m}_{Y_1} h_6 + \dot{m}_{Y_2} + Y_3 h_2$$

$$h_3 = Y_1 h_6 + (Y_2 + Y_3) h_2$$

$$h_3 = Y_1 h_6 + h_2 Y_2 + h_2 Y_3$$

$$h_3 = Y_1 h_6 + h_2 Y_2 + h_2 (1 - Y_1 - Y_2)$$

$$h_3 = Y_1 h_6 + h_2 Y_2 + h_2 - h_2 Y_1 - h_2 Y_2$$

$$h_3 = Y_1 h_6 + h_2 - Y_1 h_2$$

$$h_3 - h_2 = Y_1 (h_6 - h_2)$$

$$Y = \frac{h_3 - h_2}{h_6 - h_2} = \frac{(686.66 - 226.59) \text{ kJ/kg}}{(2882.77 - 226.59) \text{ kJ/kg}}$$

(g)  $Y_1 = .17$

$$Y_1 = Y_2$$

$$Y_2 = .17$$

AND  $Y_3 = 1 - Y_1 - Y_2 = 1 - .17 - .17$   
 $Y_3 = .66$

$$\epsilon_u = \frac{\dot{W}_{net} + \dot{Q}_p}{\dot{Q}_{in}} = \frac{\dot{W}_{net} + \dot{Q}_p}{\dot{Q}_{in}}$$

$$\dot{W}_{net} = \dot{W}_{TURS} - \dot{W}_{pump1} - \dot{W}_{pump2}$$

$$\dot{W}_{TURS} = (h_5 - h_6) + (1 - Y_1)(h_6 - h_7) + (1 - Y_1 - Y_2)(h_7 - h_8)$$

$$\dot{W}_{TURS} = ((3428.9 - 2882.77) + (1 - .17)(2882.77 - 2581.83) + (1 - .17 - .17)(2581.83 - 2242.4)) \text{ kJ/kg}$$

$$\dot{W}_{TURS} = (546.13 + 249.78 + 224.02) \text{ kJ/kg}$$

(C)  $\dot{W}_{TURS} = 1019.93 \text{ kJ/kg}$   $\dot{W}_{TURS} = 50292.75 \text{ kW}$

$$W_{\text{pump1}} = (h_2 - h_1)(1 - Y_1) = (1 - .17)(226.59 - 225.94) \text{ kJ/kg}$$

$$W_{\text{pump1}} = .54 \text{ kJ/kg}$$

$$W_{\text{pump2}} = (h_4 - h_3) = (692 - 686.66) \text{ kJ/kg}$$

$$W_{\text{pump2}} = 5.34 \text{ kJ/kg}$$

$$W_{\text{net}} = (1019.93 - .54 - 5.34) \text{ kJ/kg}$$

$$W_{\text{net}} = 1014.05 \text{ kJ/kg}$$

$$q_p = (h_7 - h_9)Y_2 = (.17)(2581.83 - 467.13) \text{ kJ/kg}$$

$$q_p = 359.5 \text{ kJ/kg}$$

$$q_{\text{in}} = h_5 - h_4 = (3428.9 - 692) \text{ kJ/kg}$$

$$q_{\text{in}} = 2736.9 \text{ kJ/kg}$$

$$E_u = \frac{W_{\text{net}} + q_p}{q_{\text{in}}} = \frac{1014.05 \text{ kJ/kg} + 359.5 \text{ kJ/kg}}{2736.9 \text{ kJ/kg}}$$

(a)  $E_u = .502$

$$\dot{W}_{\text{net}} = \dot{m} W_{\text{net}}$$

$$50000 \text{ kW} = \dot{m} (1014.05 \text{ kJ/kg})$$

(d)  $\dot{m} = 49.31 \text{ kg/s}$



$$\dot{Q}_p = \dot{m} q_p$$

$$\dot{Q}_p = 49.3 \text{ kg/s} (359.5 \text{ kJ/kg})$$

$$(e) \quad \dot{Q}_p = 17725.95 \text{ kW}$$

$$q_{out} = h_8(1 - y_1 - y_2) + h_{10}(y_1) - h_1(1 - y_1)$$

$$q_{out} = (2242.4(.66) + 467.13(.17) - 225.94(.83)) \text{ kJ/kg}$$

$$(h) \quad q_{out} = 1371.86 \text{ kJ/kg}$$

$$\dot{Q}_{out} = \dot{m} q_{out} = (49.31 \text{ kg/s})(1371.86 \text{ kJ/kg})$$

$$\dot{Q}_{out} = 67646.7 \text{ kW}$$

8. SUMMARY: AFTER DETERMINING THAT THE SYSTEM HAD AN OPEN FWH, ANALYSIS WAS STRAIGHT FORWARD. ALSO, STUDENT WAS REQUIRED TO SURMIZE THAT 2 PUMPS WERE NEEDED IN THE SYSTEM TO PROVIDE PROPER PRESSURE TO THE BOILERS.

9. MATERIALS: MATERIALS WERE NOT A CONSIDERAN FACTOR IN THIS TEST.

10. ANALYSIS,: HAD THE STUDENT INCORRECTLY ASSUMES A CFWIT, THE PROBLEM WOULD BECOME MUCH MORE DIFFICULT. THE STUDENT WAS REQUIRED TO NOTICE THAT 2 PUMPS WOULD BE REQUIRED W/ AN OFWIT, ALSO, UNDERSTANDING THAT A THROTTLE VALVE IS ISENTROPIC IS ALSO CRUCIAL TO SOLVING THIS TEST.