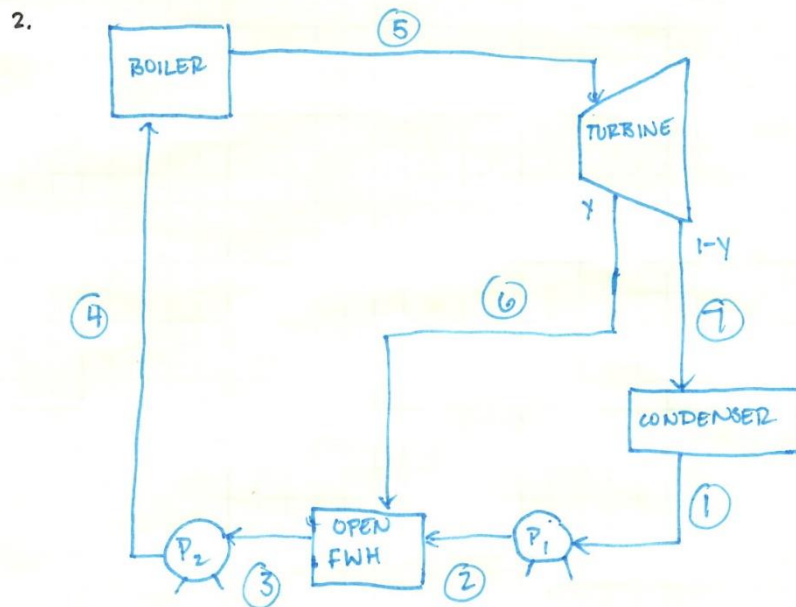
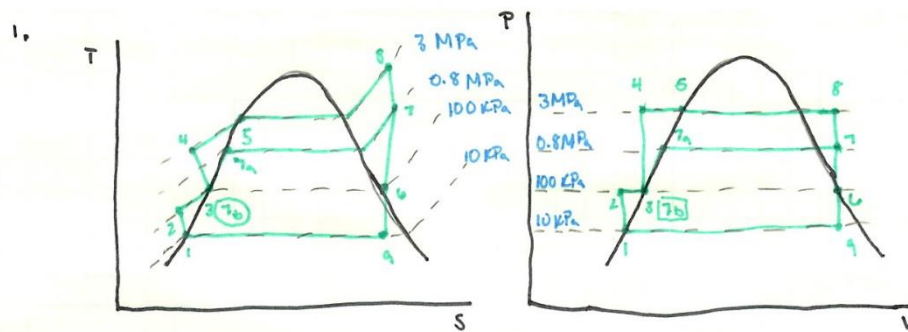


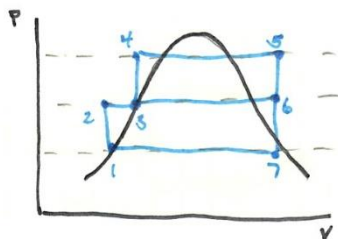
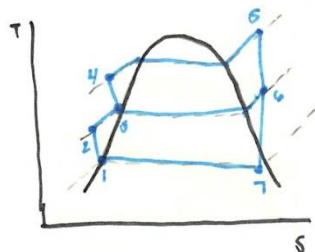
MICHAEL DELACRUZ
MET 350
TEST 2
PROFESSOR AYALA

PURPOSE: CALCULATE THE THERMAL EFFICIENCY + SPECIFIC NET WORK OF THE:
1. FULLY OPERATIONAL POWER PLANT
2. POWER PLANT WITH A MALFUNCTIONING CLOSED FEEDWATER HEATER + SUBSEQUENT LINES

SOURCES: CENGEL + BOLES. "THERMODYNAMICS - AN ENGINEERING APPROACH"
8TH EDITION, MCGRAW HILL. 2015

DRAWINGS + DIAGRAMS:





DESIGN CONSIDERATIONS:

- ① WATER IS PURE
- ② NO HEAT LOSSES IN CONNECTIONS, PIPES
- ③ NO FRICTION LOSSES FROM FLUID FLOW

DATA + VARIABLES

$$P_{\text{TURB INLET}} = 3 \text{ MPa}$$

$$T_{\text{TURB INLET}} = 400^\circ\text{C}$$

$$P_{\text{TURB EXIT}} = 10 \text{ kPa}$$

$$P_{\text{1ST EXTRACTION}} = 0.8 \text{ MPa}$$

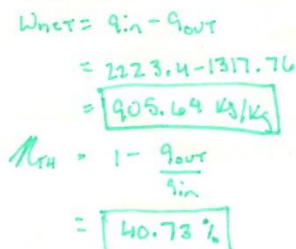
$$P_{\text{2ND EXTRACTION}} = 100 \text{ kPa}$$

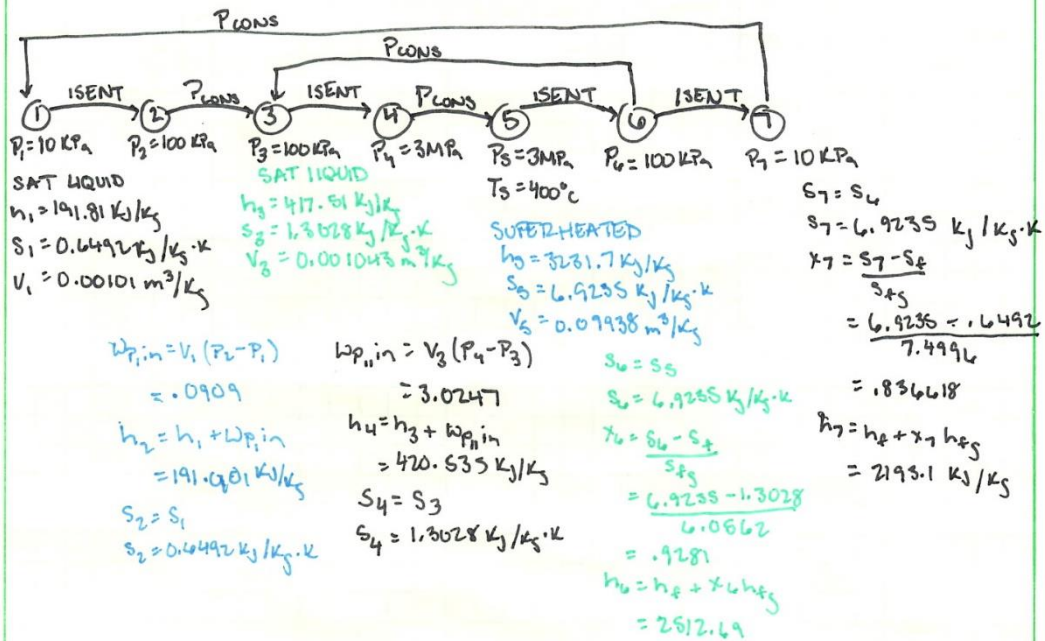
MATERIALS:

WATER

PROCEDURE:

1. FOCUS ON PART 1
2. FIND ALL STATES
3. FIND Q_{in} + Q_{out}
4. DETERMINE η_{th} + W_{net}
5. REPEAT FOR PART 2





$$y = \frac{\dot{m}_u}{\dot{m}_s}$$

$$\begin{aligned}
 \dot{q}_{in} &= h_5 - h_4 \\
 &= 3231.7 - 420.535 \\
 &= 2811.165
 \end{aligned}$$

$$\begin{aligned}
 y &= \frac{h_3 - h_2}{h_u - h_3} \\
 &= \frac{417.51 - 191.901}{2512.69 - 417.51} \\
 &= 0.10768
 \end{aligned}$$

$$\begin{aligned}
 \dot{q}_{out} &= (1-y)(h_7 - h_1) \\
 &= 0.89231(2193.1 - 191.81) \\
 &= 1785.79 \text{ kJ/kg}
 \end{aligned}$$

$$\begin{aligned}
 w_{net} &= \dot{q}_{in} - \dot{q}_{out} \\
 &= 2811.165 - 1787.79 \\
 &= 1023.37 \text{ kJ/kg}
 \end{aligned}$$

$$\begin{aligned}
 \eta_{TH} &= 1 - \frac{\dot{q}_{out}}{\dot{q}_{in}} \\
 &= 36.4\%
 \end{aligned}$$

SUMMARY:

	FULLY FUNCTIONAL	MALFUNCTION
W_{NET}	906.64 KJ/Kg	1025.37 KJ/Kg
η_{TH}	40.73 %	36.4%

ANALYSIS:

THE MALFUNCTIONING SYSTEM WAS LESS EFFICIENT THAN THE FULLY FUNCTIONAL SYSTEM. IT HELPS TO PREHEAT THE WATER BEFORE ENTERING THE BOILER. IT ALLOWS THE SYSTEM TO BECOME MORE EFFICIENT AS IT ~~PREHEATS THE WATER BEFORE ENTERING THE BOILER~~ DOES NOT MIX.