

Homework #2.1

Ch 10 Minor Losses

Ch 11 Series Pipeline Systems

MET 330 Virginia Beach Distance Learning WC2 and
Campus

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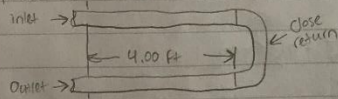
Homework 2.1

From the problems in chapter 10 minor losses and chapter 11 series pipeline systems I learned there will always be energy losses due to friction in a pipe were a fluids flows. We use Darcy-Weisbach equation to solve for major energy losses due to friction. To find friction factor we must calculate Reynolds number. The resistance coefficient K is found in the book. Sometimes problems must be iterated in order to determine the required pipe diameter.

Chapter 10: 20, 37, 39, 43, 46, 48

- 80) Determine the energy loss for a sudden contraction from a DN 125 schedule 80 steel pipe to a DN 50 schedule 80 pipe for a flow rate of 500 L/min. DN 125 $Out_o = 141.3 \text{ mm}$ $Wall_t = 9.5 \text{ mm}$ $In_o = 128.2 \text{ mm}$
 DN 50 $Out_o = 60.3 \text{ mm}$ $Wall_t = 5.54 \text{ mm}$ $In_o = 52.5 \text{ mm}$
 $Q = Av$ $V = \frac{Q}{A}$ $\frac{D_1}{D_2} = \frac{128.2}{52.5} = 2.44$ $A_2 = \frac{\pi}{4} (0.0525)^2 = 2.165 \times 10^{-3} \text{ m}^2$
 $V_2 = \frac{500 \text{ L/min}}{2.165 \times 10^{-3} \text{ m}^2} \times \frac{1 \text{ m}^3}{1000 \text{ L}} \times \frac{1 \text{ min}}{60 \text{ s}} = 3.85 \text{ m/s}$ From table $K = 0.39$
 $h_L = K \times \left(\frac{V_2^2}{2g} \right) = (0.39) \left(\frac{3.85^2 \text{ m/s}^2}{2 \times 9.81 \text{ m/s}^2} \right) = 0.295 \text{ m}$

- 37) A heat exchanger is made of two 1/2 in schedule 40 steel pipes. Compute the pressure difference between inlet and outlet. Flow rate = 12.5 gal/min ethylene glycol 77°F



$$\frac{P_1}{\rho} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\rho} + \frac{V_2^2}{2g} + Z_2 + h_f + h_L$$

$$P_1 - P_2 = \rho(h_L + h_f)$$

$$h_L = K \left(\frac{V_2^2}{2g} \right) \quad h_f = f \times \frac{L}{D} \times \frac{V_2^2}{2g}$$

$$D_{nominal} = 0.0518 \text{ ft} \quad A = 0.002 \text{ ft}^2 \quad Q = 12.5 \frac{\text{gal}}{\text{min}} \times \frac{0.1337 \text{ ft}^3}{1 \text{ gal}} \times \frac{1 \text{ min}}{60 \text{ s}} = 0.0278 \text{ ft}^3/\text{s}$$

$$V = \frac{Q}{A} = \frac{0.0278}{0.002} = 13.9 \text{ ft/s} \quad \text{Closed box coefficient} = K = f \left(\frac{L}{D} \right) \quad \frac{L}{D} = 50$$

$$\frac{D}{\epsilon} = \frac{0.0518}{1.5 \times 10^{-4}} = 345.33 \quad k = (0.026)(50) = 1.3$$

$$h_L = 1.3 \times \frac{(13.9)^2}{2 \times 32.2} = 3.9 \text{ ft} \quad N_R = \frac{VD}{\nu} = \frac{13.9 \times 0.0518}{1.54 \times 10^{-4}} = 4528.4$$

$$h_f = 0.038 \times \frac{8}{0.0518} \times \frac{13.9}{2 \times 32.2} = 17.6 \text{ ft}$$

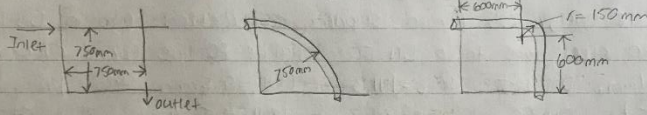
$$P_1 - P_2 = \rho(h_L + h_f)$$

$$= 68.47(3.9 + 17.6) \times \frac{1}{144} = 10.2 \text{ psi}$$

- 39) A pipe system for a pump contains a tee to permit the pressure at the outlet of the pump to be measured. Compute energy loss as $0.40 \text{ ft}^2/\text{s}$ of water at 50°F flows through the tee.

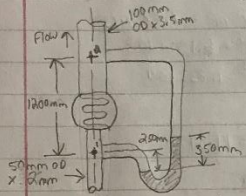
3 in schedule 40 $A = 0.0513 \text{ ft}^2$
 $V = \frac{Q}{A} = \frac{0.40 \text{ ft}^2/\text{s}}{0.0513 \text{ ft}^2} = 7.794 \text{ ft/s}$
 $\frac{L_e}{D} = 20 \quad f = 0.017 \quad h_L = f \cdot \frac{L_e}{D} \cdot \frac{V^2}{2g} = (0.017)(20) \cdot \frac{(7.794)^2}{2 \cdot 32.2} = 0.32 \text{ ft}$

- 43) The inlet and outlet are to be connected with a 50 mm OD \times 2.0 mm wall copper tube to carry 750 L/min of propyl alcohol at 25°C . Evaluate the two schemes with regard to the energy loss.



$d = 50 - 2 \times 2 = 46 \text{ mm}$ $A = \frac{\pi d^2}{4} = \frac{\pi (0.046 \text{ m})^2}{4} = 1.662 \times 10^{-3} \text{ m}^2$
 $V = \frac{Q}{A} = \frac{750 \text{ L/min} \cdot \frac{1 \text{ m}^3}{1000 \text{ L}} \cdot \frac{1 \text{ min}}{60 \text{ s}}}{1.662 \times 10^{-3} \text{ m}^2} = 0.0185 \text{ m/s}$ $V = \frac{0.0185 \text{ m/s}}{1.662 \times 10^{-3} \text{ m}^2} = 7.52 \text{ m/s}$
 $\frac{L}{D} = \frac{750}{46} = 16.3 \text{ m}$ $\frac{L_e}{D} = 43$ $\epsilon = 1.5 \times 10^{-6} \text{ m}$ $\eta = 1.92 \times 10^{-3} \text{ Pa}\cdot\text{s}$ $\rho = 828 \text{ kg/m}^3$
 $Re = \frac{\rho V D}{\eta} = \frac{828 \times 0.046 \times 7.52}{1.92 \times 10^{-3}} = 144493.67$ $\frac{P}{\rho} = \frac{0.046}{1.5 \times 10^{-6}} = 30666.67$
 Moody chart $f_t = 0.0095$ $f = 0.0162$ $K = f_t \left(\frac{L_e}{D} \right) = 0.0095(43) = 0.4085$
 $h_{L1} = 0.4085 \times \frac{(7.52)^2}{2 \times 9.81} = 1.177 \text{ m}$
 $\frac{L}{D} = \frac{120}{46} = 2.61$ $\frac{L_e}{D} = 13$ $h_{L2} = 0.0095 \times 13 \times \frac{(7.52)^2}{2 \times 9.81} = 0.356 \text{ m}$
 $h_{L3} = f \cdot \frac{L}{D} \cdot \frac{V^2}{2g}$ $h_{L3} = 0.0162 \times \frac{1.2}{0.046} \times \frac{(0.52)^2}{2 \times 9.81} = 1.218 \text{ m}$
 $h_L = h_{L1} + h_{L2} \rightarrow h_L = 0.356 + 1.218 = 1.574 \text{ m}$

- 46) Figure shows a test setup for determining the energy loss due to a heat exchanger. Water at 50°C is flowing vertically upward at $6.0 \times 10^{-3} \text{ m}^3/\text{s}$. Calculate the energy loss between points 1 and 2. Determine the resistance coefficient for the heat exchanger based on the velocity in the inlet tube.



$$d_1 = D_1 - 2t_1 \rightarrow d_1 = 50 - 2 \times 2 = 46 \text{ mm}$$

$$A_1 = \frac{\pi}{4} d_1^2 \rightarrow A = \frac{\pi}{4} (0.046 \text{ m})^2 = 1.662 \times 10^{-3} \text{ m}^2$$

$$d_2 = D_2 - 2t_2 \rightarrow d_2 = 100 - 2 \times 3.5 = 93 \text{ mm}$$

$$A_2 = \frac{\pi}{4} d_2^2 \rightarrow A = \frac{\pi}{4} (0.093 \text{ m})^2 = 6.8 \times 10^{-3} \text{ m}^2$$

$$V_1 = \frac{Q}{A_1} = \frac{6.0 \times 10^{-3} \text{ m}^3/\text{s}}{1.662 \times 10^{-3} \text{ m}^2} = 3.61 \text{ m/s} \quad V_2 = \frac{Q}{A_2} = \frac{6.0 \times 10^{-3} \text{ m}^3/\text{s}}{6.8 \times 10^{-3} \text{ m}^2} = 0.88 \text{ m/s}$$

$$\frac{p_1}{\rho} + \frac{V_1^2}{2g} + Z_1 = \frac{p_2}{\rho} + \frac{V_2^2}{2g} + Z_2 + h_L \rightarrow h_L = \frac{p_1 - p_2}{\rho} + \frac{V_1^2 - V_2^2}{2g} + (Z_1 - Z_2)$$

$$h_L = \frac{132.8(0.35)}{9.69} + \frac{9.69(0.88)}{9.69} + (0 + 1.2 \text{ m}) + \frac{3.61^2 - 0.88^2}{2 \times 9.81} = 5.07 \text{ m}$$

$$h_L = K \left(\frac{V^2}{2g} \right) \rightarrow 5.07 = K \left(\frac{3.61^2}{2 \times 9.81} \right) \quad K = 7.64$$

- 48) Compute the energy loss in a 90° bend in a steel tube used for a fluid power system.

The tube has a $1\frac{1}{4}$ OD and a wall thickness of 0.083 in. The mean bend radius is 3.25 in. The flow rate of hydraulic oil is 27.5 gal/min.

$$h_L = K \left(\frac{V^2}{2g} \right) \quad A = 6.0 \times 10^{-3} \text{ ft}^2 \quad D = 0.0874 \text{ ft}$$

$$\frac{C}{D} = \frac{3.25 \text{ in} \left(\frac{1 \text{ ft}}{12 \text{ in}} \right)}{0.0874 \text{ ft}} = 3.099 \quad \frac{L}{D} = 12.5 \quad \epsilon = 1.5 \times 10^{-4} \text{ ft}$$

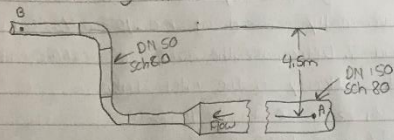
$$\frac{D}{\epsilon} = \frac{0.0874}{1.5 \times 10^{-4}} = 582.67 \quad f_t = 0.0222 \quad K = f_t \left(\frac{L}{D} \right) \rightarrow K = (0.0222)(12.5) = 0.2775$$

$$V = 27.5 \frac{\text{gal}}{\text{min}} \cdot \frac{0.1337 \text{ ft}^3}{1 \text{ gal}} \cdot \frac{1 \text{ min}}{60 \text{ sec}} = 0.06127 \text{ ft}^3/\text{s}$$

$$V = \frac{Q}{A} = \frac{0.06127}{6.0 \times 10^{-3}} = 10.21 \text{ ft/s}$$

Chapter 11 5, 13, 20

- 5) Oil is flowing at the rate of $0.015 \text{ m}^3/\text{s}$ in the system



$$A_a = \frac{\pi \times 0.1524^2}{4} = 1.822 \times 10^{-2} \text{ m}^2$$

$$A_b = \frac{\pi \times 0.0508^2}{4} = 1.905 \times 10^{-3} \text{ m}^2$$

$$V_a = \frac{0.015}{1.822 \times 10^{-2}} = 0.822 \text{ m/s}$$

$$V_b = \frac{0.015}{1.905 \times 10^{-3}} = 7.87 \text{ m/s}$$

$$N_{Re} = \frac{0.822 \times 0.1524}{2.1 \times 10^{-5}} = 6.15 \times 10^3$$

$$N_{Re} = \frac{7.87 \times 0.0508}{2.1 \times 10^{-5}} = 1.83 \times 10^4$$

$$\frac{D_a}{E} = \frac{0.1524}{4.6 \times 10^{-3}} = 3180 \quad f_{sa} = 0.035 \quad f_t = 0.019$$

$$\frac{D_b}{E} = \frac{0.0508}{4.6 \times 10^{-3}} = 1072 \quad f_{sb} = 0.028 \quad f_t = 0.019$$

$$h_1 = 0.035 \times \frac{1.822^2}{2 \times 9.81} \times \frac{0.822^2}{2 \times 9.81} = 1.75 \text{ m} \quad h_2 = 0.028 \times \frac{7.87^2}{2 \times 9.81} \times \frac{7.87^2}{2 \times 9.81} = 14.34 \text{ m}$$

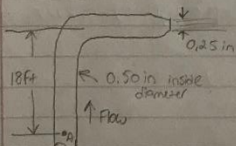
$$h_3 = 2 \times 20 \times 0.019 \times \frac{7.87^2}{2 \times 9.81} = 2.39 \text{ m} \quad h_4 = 0.37 \times \frac{7.87^2}{2 \times 9.81} = 1.17 \text{ m}$$

$$h_L = 1.75 + 14.34 + 2.39 + 1.17 = 19.65 \text{ m}$$

$$\frac{P_a}{\rho} + \frac{V_a^2}{2g} + z_a = \frac{P_b}{\rho} + \frac{V_b^2}{2g} + z_b - h_L \rightarrow P_a = \rho \left[h_L + \frac{P_b}{\rho} + (z_b - z_a) + \left(\frac{V_b^2}{2g} - \frac{V_a^2}{2g} \right) \right]$$

$$P_a = 8.80 \left[19.65 + \frac{12.5 \times 10^6}{8.80} + 4.5 + \left(\frac{0.822^2}{2 \times 9.81} - \frac{7.87^2}{2 \times 9.81} \right) \right] = 12.7 \text{ MPa}$$

- 13) Determine the velocity of flow from the nozzle if the pressure at the bottom is a) 20 psig and b) 80 psig. The nozzle has a loss coefficient K of 0.15 based on the outlet velocity head. The tube ID of 0.50 in. The 90° bend has a radius of 6.0 in. The total length of straight tube is 20.0 ft. Water at 100°F



$$A = \frac{\pi \times 0.5^2}{4} = \pi \times \left(\frac{0.5}{4} \right)^2 = 0.0013635 \text{ ft}^2$$

$$\frac{D}{E} = \frac{0.5}{1.5 \times 10^{-4}} = 2778$$

$$\frac{P_a}{\rho} + \frac{V_a^2}{2g} + z_a = \frac{P_b}{\rho} + \frac{V_b^2}{2g} + z_b + h_L$$

$$h_L = \left(\frac{P_b - P_a}{\rho} + \frac{V_b^2 - V_a^2}{2g} \right) + z_b - z_a = \left(\frac{28.8 \times 10^3}{62.4} + (0 - 18) \right) = 28.15 \text{ ft}$$

$$Q = -2.22 \times 10^{-3} \times \frac{28.15}{\log \left(\frac{3.7 \times 2778}{1.48 \times 10^{-6}} + \frac{1.78 \times 10^{-6}}{0.0127} \right)} = 0.0425 \text{ ft}^3/\text{s}$$

$$Q = -2.22 \times 0.041667 \times \frac{32.2 \times 0.0127 \times 28.15}{20} \log \left(\frac{3.7 \times 2778}{1.48 \times 10^{-6}} + \frac{1.78 \times 10^{-6}}{0.0127} \right) = 0.0425 \text{ ft}^3/\text{s}$$

$$V = \frac{Q}{A} = \frac{0.0425}{0.0013635} = 31.2 \text{ ft/s}$$

$$h_L = \left(\frac{P_b - P_a}{\rho} + \frac{V_b^2 - V_a^2}{2g} \right) + z_b - z_a = 166.615 \text{ ft}$$

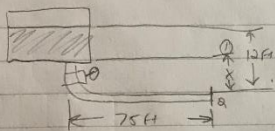
$$Q = -2.22 \times 0.041667 \times \frac{32.2 \times 0.0127 \times 166.615}{20} \log \left(\frac{3.7 \times 2778}{1.48 \times 10^{-6}} + \frac{1.78 \times 10^{-6}}{0.0127} \right) = 0.088 \text{ ft}^3/\text{s}$$

$$Q = 0.088 \text{ ft}^3/\text{s}$$

$$V = \frac{Q}{A} = \frac{0.088}{0.0013635} = 64.5 \text{ ft/s}$$

Zach Hollifield

- 20) The tank is to be drained to a sewer. Determine the size of new schedule 40 steel pipe that will carry at least 400 gal/min of water at 80°F through the system shown. The total length of pipe is 75 ft



$$\frac{P_1}{\rho} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\rho} + \frac{V_2^2}{2g} + Z_2 + h_{L1-2}$$

$$\frac{P_1 - P_2}{\rho} - Z_2 = f \frac{L}{D} \frac{V^2}{2g} + K_{valve} \frac{V^2}{2g}$$

$$\frac{P_1 - P_2}{\rho} - Z_2 = f \frac{2LQ^2}{g\pi^2 D^5} + K_{valve} \frac{2Q^2}{g\pi^2 D^4}$$

$$h_1 + Z_1 - h_L = h_2 + Z_2 \rightarrow (12 \text{ ft}) + Z - h_L = 0 + 0 = 12 \text{ ft}$$

$\epsilon = 1.5 \times 10^{-4} \text{ ft}$ Water at 80°F $\nu = 9.15 \times 10^{-6} \text{ ft}^2/\text{s}$

$$D = 0.66 \left[(1.5 \times 10^{-4})^{1.25} \left(75 \text{ ft} \times \left(\frac{400 \text{ gal/min} \times 1.488}{4.49 \text{ gal/min}} \right)^{1.485} \right) + \left(9.15 \times 10^{-6} \frac{\text{ft}^2}{\text{s}} \right) \left(\frac{400 \text{ gal/min} \times 1.488}{4.49 \text{ gal/min}} \right)^{1.485} \right]^{0.045}$$

$$\times \left(\frac{75 \text{ ft}}{32.2 \text{ ft/s}^2 \times 12 \text{ ft}} \right)^{0.22} = 0.26 \text{ ft}$$

HW 2.1

Chapter 10: 20, 37, 39, 43, 46, 49

Chapter 11: 5, 13, 20

- 1 We learned about series Pipeline systems and the different types of classes for them. Class I determines energy losses, Class II determines flow rate, and Class III determines pipe diameter. From chapter 10, we learned that there will always be energy losses due to friction in a pipe. We must calculate Reynolds number to find friction factor.

	OD	thick	ID
10.20 DN 125 sch 80	141.3 mm	9.5	122.3 mm - 0.1223
DN 50 sch 80	60.3 mm	5.54	49.3 mm - 0.0493
$Q = 500 \text{ L/min} = 0.0083 \text{ m}^3/\text{s}$			

$$h_L = K (V_2^2 / 2g)$$

$$\frac{D_1}{D_2} = \frac{0.1223}{0.0493} = 2.480$$

$$V_2 = \frac{Q}{A_2} = \frac{0.0083 \text{ m}^3/\text{s}}{1.905 \times 10^{-3} \text{ m}^2} = 4.35 \text{ m/s}$$

$$K = 0.38$$

$$h_L = 0.38 (4.35^2 / 2(9.81)) = 0.366 \text{ m}$$

10.37 $\frac{1}{2}$ SCH 40 pipe — $\frac{ID}{4} = 0.0519 \text{ ft}$ $\frac{A}{4} = 0.00211$

Ethylene glycol at 77°F — $\rho = 213$ $k = 3.39 \times 10^{-4}$ $\epsilon = 6.8 \times 10^{-5}$

$Q = 12.5 \text{ gal/min} = 0.0278 \text{ ft}^3/\text{s}$

$L = 4 \text{ ft} \times 2$

$50 \text{ ft}/\text{s}$

$V = \frac{Q}{A} = \frac{0.0278}{0.00211} = 13.175 \text{ ft/s}$

$Re = \frac{\rho V D}{\mu} = \frac{213 \cdot 13.175 \cdot 0.0519}{3.39 \times 10^{-4}} = 4300.74$

$f = 0.0429$ — Excel $D = 0.0519$ $\epsilon = 0.000015$ $Re = 4300.74$

$h = 50 \cdot 0.0429 = 2.145$

$h_{\text{valve}} = K \left(\frac{V^2}{2g} \right) = 2.145 \cdot \frac{13.175^2}{64.4} = 5.76 \text{ ft}$

$h_{\text{pipe}} = 0.0429 \left(\frac{L}{D} \right) \left(\frac{13.175^2}{64.4} \right) = 17.96 \text{ ft}$

$h_{\text{total}} = 5.76 + 17.96 = 23.46 \text{ ft}$ $\frac{P_1}{\gamma} + h_m + \frac{V_1^2}{2g} = \frac{P_2}{\gamma} + h_{\text{total}}$

$P_1 - P_2 = 23.64 \cdot 62.4 = 1618.63 \frac{\text{lb}}{\text{ft}^2} = \boxed{11.24 \text{ psi}}$

10.39 $Q = 0.40 \text{ ft}^3/\text{s}$

$H_2O @ 50^\circ F = \gamma = 62.4 \quad \nu = 1.40 \times 10^{-5}$

Std SCH 40 $ID = 0.2557 \quad A = 0.05132 \text{ ft}^2$

$$V = \frac{Q}{A} = \frac{0.40 \text{ ft}^3/\text{s}}{0.05132 \text{ ft}^2} = 7.79 \text{ ft/s}$$

$$N_R = \frac{VD}{\nu} = \frac{7.79 \cdot 0.2557}{1.40 \times 10^{-5}} = 142,279$$

$f = 0.020$ - Excel $D = 0.2557 \quad \epsilon = 1.5 \times 10^{-4} \quad N_R = 142,279$

$k = 20$ for tee

$L = 20 \cdot 0.00 = 0.4$

$$h_L = K \frac{V^2}{2g} = 0.4 \left[\frac{7.79^2}{64.4} \right] = 0.377 \text{ ft}$$

10.43 Copper Tube 50 mm OD 2 mm wt

$1.5 \times 10^{-6} = \epsilon \quad ID = 46 \text{ mm}$

$Q = 250 \text{ L/min} = 0.0125 \text{ m}^3/\text{s}$

Propyl alcohol @ $25^\circ C$

$\gamma = 7.87$

$\rho = 802$

$\nu = 1.92 \times 10^{-5}$

$$h_{\text{total}} = f \frac{L}{D} \frac{V^2}{2g} + V_{\text{elbow}} \frac{V^2}{2g}$$

$$h_{\text{total}} = \left[0.0164 \cdot \left(\frac{1.2}{0.046} \right) \cdot \frac{7.52^2}{2g} \right] + \left[0.1235 \left(\frac{7.52^2}{19.62} \right) \right]$$

$$= 1.619 \text{ m}$$

10.46 Water @ 50°C - $\gamma = 9.69$
 $\delta_{hg} = 132.8$

$$Q = 6.0 \times 10^{-3} \text{ m}^3/\text{s}$$

	ID	A
$P_1 = 50 \text{ mm OD w/ 2 mm wall}$	46 mm	$1.662 \times 10^{-3} \text{ m}^2$

$P_2 = 100 \text{ mm OD w/ 3.5 mm wall}$	93 mm	$6.793 \times 10^{-3} \text{ m}^2$
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$$V_1 = \frac{Q}{A_1} = \frac{6 \times 10^{-3}}{1.662 \times 10^{-3}} = 3.61 \text{ m/s}$$

$$V_2 = \frac{Q}{A_2} = \frac{6.0 \times 10^{-3}}{6.793 \times 10^{-3}}$$

$$= 0.898 \text{ m/s}$$

$$10.43 \text{ cm} \quad v = \frac{Q}{A} = \frac{0.0175}{1.462 \times 10^{-3}} = 7.52 \text{ m/s}$$

$$Na = \frac{P \cdot v \cdot D}{n} = \frac{802 \cdot 7.52 \cdot 0.046}{1.92 \times 10^{-3}} = 144,494$$

$$f = 0.0166$$

Scheme I

$$\frac{r}{D} = \frac{0.250}{0.046} = 16.7$$

$$\frac{D}{L} = \frac{0.046}{1.5 \times 10^{-6}} = 30,666.7$$

$$f \text{ from diagram } 0.7 = 0.0095$$

$$\frac{L}{D} \text{ from } 10.6 = 43$$

$$K = 43 \cdot 0.0095 = 0.4085$$

$$h = K \frac{v^2}{2g} \quad h = \frac{0.4085 \cdot 7.52^2}{19.62} = 1.177 \text{ m}$$

Scheme II

$$\frac{r}{D} = \frac{0.150}{0.046} = 3.26$$

$$\frac{L}{D} \text{ from } 10.6 = 13 \quad K = 13 \cdot 0.0095 = 0.1235$$

10.48 $Q = 27.5 \text{ gal/min} = 0.0613 \text{ ft}^3/\text{s}$
 $r = 3.25$

90° Bend steel wire with OD of $1\frac{1}{4}$ and WE of 0.093 in

ID A
 $0.0903 \text{ ft} = 1.084 \text{ in}$ 6.41×10^{-3}

$$V = \frac{Q}{A} = \frac{0.0613}{6.41 \times 10^{-3}} = 9.56 \text{ ft/s}$$

$$\frac{L}{D} = \frac{3.25}{1.084} = 3.0$$

From Figure 10.24 $\frac{L}{D} = 3$

$$\frac{Q}{\xi} = \frac{0.0903}{1.5 \times 10^{-4}} = 602 \quad \text{from Diagram } F = 0.0225$$

$$K = \left(\frac{L}{D} \right) F = 3 \cdot 0.0225 = 0.2425$$

$$h_L = K \frac{V^2}{2g} = h_L = 0.2425 \cdot \frac{7.52^2}{64.4} = 0.415 \text{ ft}$$

manometer

$$P_1 + ((0.125)(9.69)) - ((0.350)(132.8)) - ((1.1)(9.89)) = P_2$$

$$P_1 - P_2 = ((0.35 \cdot 132.8) + (0.85 \cdot 9.69))$$

$$P_1 - P_2 = 54.7165$$

$$\frac{P_1 - P_2}{\gamma} = 5.647 \text{ m}$$

$$\frac{P_1}{\gamma} + z_1 + \frac{V_1^2}{2g} = \frac{P_2}{\gamma} + z_2 + \frac{V_2^2}{2g} + h_L$$

$$h_L = \frac{P_1 - P_2}{\gamma} + (z_1 - z_2) + \frac{V_1^2 - V_2^2}{2g}$$

$$h_L = 5.647 - 1.2 + \frac{3.61^2 - 0.483^2}{2g}$$

$$h_L = 5.07 \text{ m}$$

$$h_L = K \frac{V^2}{2g} = 5.07 = K \frac{3.61^2}{19.61}$$

$$K = 7.63$$

$$h_{\text{entr}} = 2 K \frac{V^2}{2g} = 2(0.39 \cdot \frac{7.874^2}{19.62}) = 2.46 \text{ m}$$

$$h_{\text{pipe 1}} = f \cdot \frac{L}{D} \cdot \frac{V^2}{2g}$$

$$N_R = \frac{VD}{\nu} = \frac{0.992 \cdot 0.1463}{2.12 \times 10^{-5}} = 6155.64$$

$$f = 0.0360 \text{ steel}$$

$$h_{\text{L}} = 0.0360 \cdot \left(\frac{150}{0.1463} \right) \cdot \left(\frac{0.992^2}{19.62} \right) = 1.80 \text{ m}$$

$$h_{\text{pipe 2}} = f \cdot \frac{L}{D} \cdot \frac{V^2}{2g}$$

$$N_R = \frac{7.874 \cdot 0.0493}{2.12 \times 10^{-5}} = 18,310.74$$

$$f = 0.0285 \text{ steel}$$

$$h_{\text{pipe 2}} = 0.0285 \left(\frac{46}{0.0493} \right) \left(\frac{7.874^2}{19.62} \right) = 14.61 \text{ m}$$

$$h_{\text{total}} = 14.61 + 1.80 + 2.46 + 1.20 = 20.07$$

$$\frac{P_1}{\gamma} + z_1 + \frac{V_1^2}{2g} = \frac{P_2}{\gamma} + z_2 + \frac{V_2^2}{2g} + h_{\text{L}}$$

$$\frac{P_1}{8.8} + \frac{0.992^2}{19.62} = \frac{12.5 \times 10^3}{8.8} + 4.15 + \frac{7.874^2}{19.62} + 20.07$$

$$11.5 \quad Q_1 = 0.015 \text{ m}^3/\text{s}$$

$$\gamma = 8.80 \text{ kN/m}^3$$

$$V = 2.12 \times 10^{-5} \text{ m}^2/\text{s}$$

$$L \text{ of DN 150 pipe} = 150 \text{ m}$$

$$L \text{ of DN 50 pipe} = 8 \text{ m}$$

$$P_0 = 12.5 \text{ MPa} = 12.5 \times 10^3 \frac{\text{kN}}{\text{m}^2}$$

$$V_1 = \frac{Q}{A_1} = \frac{0.015}{1.482 \times 10^{-2}} = 0.992 \text{ m/s}$$

$$V_2 = \frac{Q}{A_2} = \frac{0.015}{1.905 \times 10^{-3}} = 7.874 \text{ m/s}$$

Contraction

$$\frac{D_1}{D_2} = \frac{0.1463}{0.0493} = 2.97$$

$$10.4, K = 0.38$$

$$h_L = K \frac{V^2}{2g} = 0.38 \cdot \frac{7.874^2}{19.62} = 1.20$$

2 90° Elbows

$$\frac{D}{L} = \frac{0.0493}{4.6 \times 10^{-5}} = 1071.74$$

$$f + \text{Figure 8.7} = 0.0195$$

$$K = \frac{L}{D} \cdot f + = 0.0195 \times 20 = 0.39$$

$$\frac{P_1}{\rho} = 1449.14$$

$$P_1 = 12743.63 \text{ kPa}$$

$$12.74 \text{ MPa}$$

$$11.13 \text{ } D = 0.50 \text{ in}$$

$$\frac{Q}{L} = \frac{0.5}{1.5 \cdot 12 \cdot 10^{-4}} = 277.777$$

$$h_L = \frac{P_1 - P_2}{\gamma} + \frac{V_A^2}{2g} - \frac{V_B^2}{2g} \quad (P_A = P_B)$$

$$\frac{2880}{62.4} = 46.15$$

$$= 26.15 \text{ ft}$$

$$Q = -2.22 \cdot 10^2 \sqrt{\frac{g \cdot D \cdot h_L}{L}} \log \left[\frac{1}{3.7 D/\epsilon} + \frac{1.764 V}{D \sqrt{g \cdot D \cdot h_L}} \right]$$

$$= -2.22 \cdot 10^2 \cdot 0.041667^2 \sqrt{\frac{32.2 \cdot 0.041667 \cdot 26.15}{20}} \log \left(\frac{1}{3.7 \cdot 277.777} + \frac{1.764 \cdot 7.3}{0.041667 \sqrt{32.2 \cdot 0.041667 \cdot 26.15}} \right)$$

$$0.041667$$

$$\sqrt{32.2 \cdot 0.041667 \cdot 26.15}$$

$$26$$

$$= 0.0425 \text{ ft}^3/\text{s}$$

$$V = \frac{Q}{A} = \frac{0.056}{0.0013635}$$

$$= 41.53 \text{ ft/s}$$

$$11.20 \quad \frac{P_1}{\gamma} + z_1 + \frac{V_1^2}{2g} - h_L = \frac{P_2}{\gamma} + z_2 + \frac{V_2^2}{2g}$$

$$h_1 + z_1 - h_L = h_2 + z_2$$

$$12 - x + x - h_L = 0$$

$$h_L = 12 \text{ ft}$$

$$z = 1.5 \times 10^{-4} \text{ ft from table}$$

$$V = 9.15 \times 10^{-6} \text{ ft}^2/\text{s} @ 50^\circ \text{F}$$

$$D = 0.66 \left[9^{1.75} \left(\frac{LQ^2}{gh_L} \right)^{4.75} + 1.49 \left(\frac{L}{gh_L} \right)^{9.75} \right]^{0.04}$$

$$D = 0.66 \left[(1.5 \times 10^{-4})^{1.75} \left[\frac{75 \text{ ft} \cdot 400 \frac{\text{gal}}{\text{min}} \cdot 1 \text{ ft}^2/\text{s}}{4.49 \frac{\text{gal}}{\text{min}}} \right]^2 \right]^{4.75} \right]^{0.04}$$

$$+ \left[\frac{9.15 \cdot 10^{-6} \text{ ft}^2}{5} \cdot \frac{400 \frac{\text{gal}}{\text{min}} \cdot 1 \text{ ft}^2/\text{s}}{4.49 \frac{\text{gal}}{\text{min}}} \right]^{9.75} \left[\frac{75 \text{ ft}}{32.2 \frac{\text{ft}}{\text{s}^2}} \right]^2$$

$$D = 0.66 \cdot (2.295 \cdot 10^{-9}) + (6.12 \cdot 10^{-10}) \right]^{0.04} = 0.3006 \text{ ft}$$

$$V = \frac{Q}{A} = \frac{0.04250}{0.0013635}$$

$$= 31.17 \text{ ft/s}$$

$$B) h_L = \frac{P_1 - P_2}{\gamma} + \frac{V_A^2}{2g} - \frac{V_B^2}{2g} + z_A - z_B$$

$$\frac{115.20}{62.4} +$$

$$= 166.615 \text{ ft}$$

$$Q = -2.22 \cdot 0.041667^2 \sqrt{\frac{32.2 \cdot 0.041667 \cdot 166.61}{20}} \log \left[\frac{5.7477778 + 1.784737 \times 10^{-5}}{0.041667 \sqrt{32.2 \cdot 0.041667 \cdot 166.61}} \right]$$

$$= 0.046 \text{ ft}^3/\text{s}$$

What we learn in class

- 1) We learned that series pipeline systems and how it worked in equations. We have different classes (I and II) to explain the flow rate and type of pipe diameter we use. We use the Darcy-Weisbach equation to help solve energy losses. We had to look for certain values like friction in the back of the book. Problems are sometimes iterated to find the pipe diameter.

Nathaniel
Rypligen

CH 10 2012 75 99, 43, 44 pg (14.11, 5, 13, 22)

2d. Determine the energy loss for a sudden contraction
from a DN 125 schedule 80 steel to a DN 50 schedule
80 pipe for a flow rate of 500 L/min

$K = \text{table 10.3}$

Table 10.3: $D_1 = 122.3 \text{ mm}$ $A_1 = 1.23 \times 10^{-2} \text{ m}^2$
 $D_2 = 49.3 \text{ mm}$ $A_2 = 1.905 \times 10^{-3} \text{ m}^2$

Use A_1 A_2
 $\frac{A_1}{A_2} = \frac{1.23 \times 10^{-2}}{1.905 \times 10^{-3}} = 2.48$

convert

$Q = A_1 v_1 = A_2 v_2$
 $v_2 = \frac{Q}{A_2} = \frac{500 \text{ L/min}}{1.905 \times 10^{-3} \text{ m}^2} = 262467.19 \text{ gal/min} \left(\frac{1 \text{ m}^3}{60000 \text{ L}} \right)$

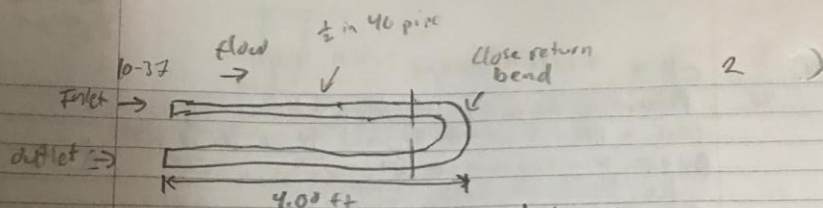
$V = 4.374 \text{ m/s}$

$K = 0.38 \text{ (table 10.3)}$

Sudden
Contraction

$h_L = K \left(\frac{v_2^2}{2g} \right) = 0.38 \left(\frac{(4.374 \text{ m/s})^2}{2(9.81 \text{ m/s}^2)} \right)$

$h_L = 0.570 \text{ m}$



compute **pressure** difference between the inlet and the outlet for a flow rate of 12.5 gal/min of ethylene glycol at 77°F

$P_1 = \text{Inlet}$ $P_2 = \text{Outlet}$ $V_1 = V_2$

Bernoulli's equation

$$\frac{P_1}{\gamma} + \frac{V_1^2}{2g} + z_1 - h_{\text{pipe}} - h_{\text{head}} = \frac{P_2}{\gamma} + \frac{V_2^2}{2g} + z_2$$

$$\frac{P_1}{\gamma} - h_{\text{pipe}} - h_{\text{head}} = \frac{P_2}{\gamma}$$

Flow $\frac{P_1}{\gamma} - \frac{P_2}{\gamma} = h_p - h_h$ $P_1 - P_2 = \gamma(h_p - h_h)$

$Q = VA$ $v = \frac{Q}{A} = \left(\frac{12.5 \text{ gal/min}}{0.00211 \text{ ft}^2} \right) \left(\frac{1 \text{ ft}^3}{7.48 \text{ gal}} \right) = 13.19 \text{ ft/s}$

$$N_R = \frac{v D_p}{\mu} = \frac{(13.19)(0.0516 \text{ ft})}{3.38 \times 10^{-3}} = 4.31 \times 10^3$$

$$\frac{D}{E} = \frac{0.0516}{1.5 \times 10^{-4}} = 345 \quad S = 0.041$$

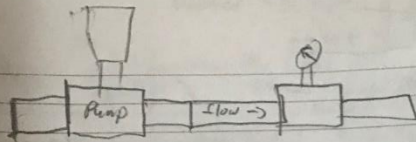
Table 10.5 $\frac{L}{D} = 50$ $S_T = 0.007$ $S = 0.044$ $h = \frac{f_T L V^2}{D \gamma}$

$$h_{\text{head}} = \frac{(0.007)(50)(13.19)^2}{2(32.2)} = 3.65 \text{ ft}$$

$$h_{\text{pipe}} = \frac{(0.041)(800)}{(0.0516)} \left(\frac{13.19^2}{2(32.2)} \right) = 17.12 \text{ ft}$$

$$P_1 - P_2 = \frac{(68.97)(17.12 + 3.65)}{1.44} = 9.87 \text{ psi}$$

10.31



3

Compute the energy loss as $0.40 \text{ ft}^3/\text{s}$
of water at 50°F flow through the ICR

$$V = \frac{Q}{A} = \frac{0.40 \text{ ft}^3/\text{s}}{0.05132 \text{ ft}^2} = 7.79 \text{ ft/s}$$

Table 10.5

$$\text{Re} = \frac{VD}{\nu} = 20$$

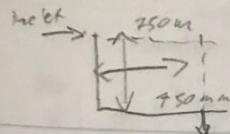
$$f = \frac{16}{\text{Re}} = 0.8$$

$$h_L = f \cdot \frac{L}{D} \cdot \frac{V^2}{2g} = (0.016)(20) \left(\frac{7.79^2}{2(32.2)} \right)$$

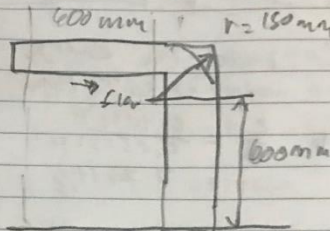
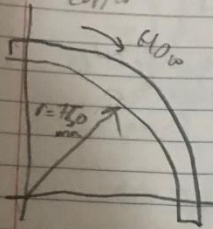
$$h_L = 0.346 \text{ ft}$$

43

4 1)



Both 50 mm OD, 20 mm wall
copper
outlet
 $P = 802$



(b)
Pipe Bend

$$V = \frac{Q}{A} = \frac{750 \text{ L/min}}{1.943 \times 10^{-2} \text{ m}^2} = \left(\frac{1 \text{ m}^3}{60000 \text{ L/min}} \right) = 6.43 \text{ m/s}$$

$$N_R = \frac{VD\rho}{\mu} = \frac{(6.43)(0.0498)(802)}{1.92 \times 10^{-3}} = 1.34 \times 10^5$$

Pipe bend

$$\frac{r}{D} = \frac{750 \text{ mm}}{49.9 \text{ mm}} = 15.1 \quad \frac{L_e}{D} = 40.5$$

$$\frac{D}{L_e} = \frac{0.0498}{1.5 \times 10^{-5}} = 33200 \quad (\text{fig 2.3})$$

Friction factor

$$f_T = 0.010$$

$$f = 0.017$$

$$h_L = f_T \frac{L_e}{D} \frac{V^2}{2g} = (0.010)(40.5) \left(\frac{(6.43)^2}{2(9.81)} \right) = 0.85 \text{ m}$$

(c)

Bend tube

$$h_{L_{\text{bend}}} = f_T \frac{L_e}{D} \frac{V^2}{2g} = (0.010)(11.8) \left(\frac{(6.43)^2}{2(9.81)} \right) = 0.125 \text{ m}$$

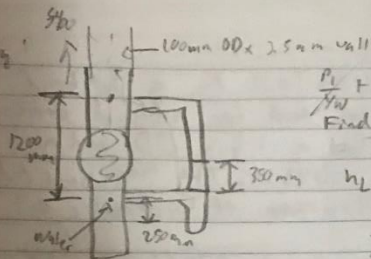
$$\frac{L_e}{D} = 11.8$$

$$\frac{r}{D} = \frac{150 \text{ mm}}{47.8 \text{ mm}} = 3.1 \text{ mm}$$

$$0.125 \text{ m} + 0.85 \text{ m} = 1.1 \text{ m}$$

Water = 50°C $\uparrow 60 \times 10^{-3} \text{ m}^3/\text{s}$

Problem 4/6



$$\frac{P_1}{\gamma_w} + z_1 + \frac{V_1^2}{2g} = \frac{P_2}{\gamma_w} + z_2 + \frac{V_2^2}{2g}$$

Find h_L

$$h_L = \frac{P_1 - P_2}{\gamma_w} + z_1 - z_2 + \frac{V_1^2 - V_2^2}{2g}$$

$$z_1 - z_2 = -1.20 \text{ m}$$

$$A_1 = \frac{\pi D_1^2}{4} = \frac{\pi (0.05 \text{ m})^2}{4} = 0.001963 \text{ m}^2$$

$$A_2 = \frac{\pi D_2^2}{4} = \frac{\pi (0.10 \text{ m})^2}{4} = 0.00785 \text{ m}^2$$

$$V_1 = \frac{Q}{A_1} = \frac{6.0 \times 10^{-3} \text{ m}^3/\text{s}}{0.001963 \text{ m}^2} = 3.056 \text{ m/s} \quad V_2 = \frac{Q}{A_2} = \frac{6.0 \times 10^{-3} \text{ m}^3/\text{s}}{0.00785 \text{ m}^2} = 0.764 \text{ m/s}$$

$$\frac{V_1^2}{2g} = \frac{(3.056 \text{ m/s})^2}{2(9.81)} = 0.476 \text{ m} \quad \frac{V_2^2}{2g} = \frac{(0.764 \text{ m/s})^2}{2(9.81)} = 0.0297 \text{ m}$$

Manometer follow based on manometer

$$P_1 + \gamma_w(0.25 \text{ m}) - \gamma_w(0.35 \text{ m}) - \gamma_w(1.10 \text{ m}) = P_2$$

$$\frac{P_1 - P_2}{\gamma_w} = \frac{\gamma_w(0.25)}{\gamma_w} - \frac{\gamma_w(0.35)}{\gamma_w} - \frac{\gamma_w(1.10)}{\gamma_w} = \frac{132.8(0.25)}{9.67 \text{ kN/m}^3} - \frac{9.69(0.85)}{9.67}$$

$$\frac{P_1 - P_2}{\gamma_w} = 4.796 + 0.85 = 5.646 \text{ m}$$

$$h_L = 5.65 - 1.20 + 0.476 - 0.0297$$

$$h_L = 4.90 \text{ m}$$

$$h_L = K \frac{V_1^2}{2g} \quad \text{Find } K$$

$$K = \frac{h_L}{\frac{V_1^2}{2g}} = \frac{4.90 \text{ m}}{0.476 \text{ m}} = 10.3$$

$$K = 10.3$$

Prob. Compute the energy loss in a 90° bend in a steel tube used for a fluid power system. The tube has a $1\frac{1}{4}$ in OD and a wall thickness of 0.083 in. The mean bend radius is 3.25 in. The flowrate of hydraulic oil

27.5 gal/min

$$A = 6.0 \times 10^{-3} \text{ ft}^2 \quad D = 0.0874 \text{ ft} \quad E = 1.5 \times 10^{-4} \text{ ft}$$

$$h_L = K \left(\frac{V}{2g} \right)$$

$$\frac{P}{D} = 3.25 \text{ in} \left(\frac{1 \text{ ft}}{12 \text{ in}} \right) = 3.099 \quad \frac{V}{D} = 12.5$$

$$\frac{D}{E} = \frac{0.0874}{1.5 \times 10^{-4}} = 602.2 \quad E_f = 0.0222$$

$$K = (0.0222)(12.5) = 0.2775$$

$$V = 27.5 \frac{\text{gal}}{\text{min}} \left(\frac{0.1337 \text{ ft}^3}{\text{gal}} \right) \left(\frac{1 \text{ min}}{60 \text{ sec}} \right)$$

$$Q = 0.06127 \text{ ft}^3/\text{s}$$

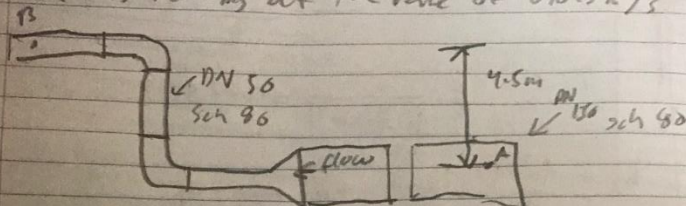
$$V = \frac{Q}{A} = \frac{0.06127}{6.0 \times 10^{-3}} = 9.70 \text{ ft/s}$$

clt II

natural
frequency

7

5) Oil is flowing at the rate of $0.015 \text{ m}^3/\text{s}$



$$A_A = \frac{\pi (0.1463)^2}{4} = 1.682 \times 10^{-2} \text{ m}^2 \quad A_B = \frac{\pi (0.0993)^2}{4} = 1.925 \times 10^{-3} \text{ m}^2$$

$$V_A = \frac{0.015}{1.682 \times 10^{-2}} = 0.892 \text{ m/s} \quad V_B = \frac{0.015}{1.925 \times 10^{-3}} = 7.87 \text{ m/s}$$

$$N_{RA} = \frac{(0.892)(0.1463)}{2.12 \times 10^{-5}} = 6.15 \times 10^3 \quad N_{RB} = \frac{7.87 \times 0.0993}{2.12 \times 10^{-5}} = 1.33 \times 10^4$$

$$D_A = \frac{0.1463}{2} = 0.07315 \text{ m}; f_A = 0.035 \frac{D_A}{\mu} = \frac{0.07315}{2 \times 9.81} = 0.0037 \text{ s}; f_B = 0.028$$

$$f_{TB} = 0.017 \quad \text{total} \quad h_L = 19.65 \text{ m}$$

$$h_1 = 0.035 \times \frac{186}{2 \times 9.81} \left(\frac{0.892^2}{2 \times 9.81} \right) = 1.73$$

$$h_2 = 0.028 \left(\frac{2}{2 \times 9.81} \right) \left(\frac{7.87^2}{2 \times 9.81} \right) = 14.34 \text{ m}$$

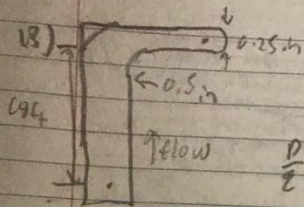
$$h_3 = 2(20) \left(\frac{0.014}{2 \times 9.81} \right) \left(\frac{7.87^2}{2 \times 9.81} \right) = 2.39 \text{ m}$$

$$h_{xy} = 0.37 \left(\frac{7.87^2}{2 \times 9.81} \right) = 1.17 \text{ m}$$

$$\frac{P_A}{\gamma} + \frac{V_A^2}{2g} + z_A = \frac{P_B}{\gamma} + \frac{V_B^2}{2g} + z_B - h_L \quad P_A = \gamma \left[h_L + \frac{V_B^2}{2g} + (z_B - z_A) \right]$$

$$P_A = 8.80 \left[19.65 + \frac{12.5 \times 10^6}{8.80} + 4.5 + \left(\frac{0.892^2}{2 \times 9.81} - \frac{7.87^2}{2 \times 9.81} \right) \right]$$

$$P_A = 12.7 \text{ MPa}$$



$$A = \frac{\pi D^2}{4} = \frac{\pi (0.25)^2}{4} = 0.0013635 \text{ m}^2$$

$$\frac{D}{L} = \frac{0.5}{(1.5)(2 \times 10^{-4})} = 277.8 \text{ s}$$

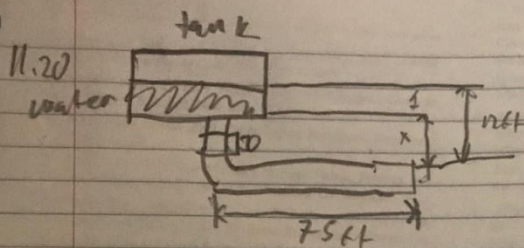
$$h_L = \frac{f_1 - f_2}{8} + \frac{v_A^2}{2g} - \frac{v_B^2}{2g} + z_A - z_B$$

$$h_L = 29.15 \text{ ft} \quad S = 332 \quad n = 0.041667 \quad L = 20$$

$$Q = -2.22 \cdot 0^2 \sqrt{\frac{S \cdot D \cdot h_L}{L}} \log \left[\frac{1.784 V}{n \sqrt{S \cdot D \cdot h_L}} \right]$$

$$Q = 0.0425 \text{ m}^3/\text{s}$$

$$V = \frac{Q}{A} = \frac{0.0425}{0.0013635} = 67.53 \text{ ft/s}$$



$$\frac{P_1}{\gamma} + \frac{V_1^2}{2g} + z_1 = \frac{P_2}{\gamma} + \frac{V_2^2}{2g} + z_2 + h_{L1-2}$$

$$\frac{P_1 - P_2}{\gamma} - 2g = \frac{8LQ^2}{\pi^2 g^2} \frac{1}{D^5} + K_{valve} \frac{8^2}{\pi^2} \frac{1}{D^4}$$

$$h_1 + z_1 - h_L = h_2 + z_2$$

$$(12 - x) + x - h_L = 0 + 0 = 12 \text{ ft}$$

$$6.15 \times 10^{-4} \text{ ft water at } 80^\circ \text{F } V = 2.15 \times 10^{-4} \text{ ft}^2/\text{s}$$

$$D = 0.66 \left((1.5 \cdot 10^{-4})^{1.25} \left(\frac{75 \left(\frac{4000 \text{ gal}}{\text{min}} \cdot \frac{1 \text{ ft}^3}{7.48 \text{ gal}} \right)^2}{449 \text{ gal/min}} \right) \right)$$

$$+ \left(\frac{2.15 \times 10^{-4} \text{ ft}^2/\text{s}}{7.48 \text{ gal/min}} \right) \left(\frac{4000 \text{ gal}}{\text{min}} \cdot \frac{1 \text{ ft}^3}{7.48 \text{ gal}} \right)^{1.4} \left(\frac{75}{32.2 \text{ ft/s}^2} \right)^{5.2}$$

$$D = 0.26 \text{ ft}$$