

**10-57** A Rankine steam cycle modified with two closed feedwater heaters is considered. The  $T$ - $s$  diagram for the ideal cycle is to be sketched. The fraction of mass extracted for the closed feedwater heater  $z$  and the cooling water flow rate are to be determined. Also, the net power output and the thermal efficiency of the plant are to be determined.

**Assumptions** 1 Steady operating conditions exist. 2 Kinetic and potential energy changes are negligible.

**Analysis** (b) Using the data from the problem statement, the enthalpies at various states are

$$h_1 = h_f @ 20 \text{ kPa} = 251 \text{ kJ/kg}$$

$$\nu_1 = \nu_f @ 20 \text{ kPa} = 0.00102 \text{ m}^3/\text{kg}$$

$$\begin{aligned} w_{p1,\text{in}} &= \nu_1 (P_2 - P_1) \\ &= (0.00102 \text{ m}^3/\text{kg})(5000 - 20 \text{ kPa}) \left( \frac{1 \text{ kJ}}{1 \text{ kPa} \cdot \text{m}^3} \right) \\ &= 5.1 \text{ kJ/kg} \end{aligned}$$

$$h_2 = h_1 + w_{p1,\text{in}} = 251 + 5.1 = 256.1 \text{ kJ/kg}$$

Also,

$$h_3 = h_{11} = h_f @ 245 \text{ kPa} = 533 \text{ kJ/kg}$$

$$h_{12} = h_{11} \quad (\text{throttle valve operation})$$

$$h_4 = h_9 = h_f @ 1400 \text{ kPa} = 830 \text{ kJ/kg}$$

$$h_{10} = h_9 \quad (\text{throttle valve operation})$$

An energy balance on the closed feedwater heater gives

$$1h_2 + zh_7 + yh_{10} = 1h_3 + (y+z)h_{11}$$

where  $z$  is the fraction of steam extracted from the low-pressure turbine. Solving for  $z$ ,

$$z = \frac{(h_3 - h_2) + y(h_{11} - h_{10})}{h_7 - h_{11}} = \frac{(533 - 256.1) + (0.1446)(533 - 830)}{2918 - 533} = \mathbf{0.09810}$$

(c) An energy balance on the condenser gives

$$\dot{m}_8 h_8 + \dot{m}_w h_{w1} + \dot{m}_{12} h_{12} = \dot{m}_1 h_1 + \dot{m}_w h_{w2}$$

$$\dot{m}_w (h_{w2} - h_{w1}) = \dot{m}_8 h_8 + \dot{m}_{12} h_{12} - \dot{m}_1 h_1$$

Solving for the mass flow rate of cooling water, and substituting with correct units,

$$\begin{aligned} \dot{m}_w &= \frac{\dot{m}_5 [(1-y-z)h_8 + (y+z)h_{12} - 1h_1]}{c_{pw} \Delta T_w} \\ &= \frac{(75) [(1 - 0.1446 - 0.09810)(2477) + (0.1446 + 0.09810)(533) - 1(251)]}{(4.18)(10)} \\ &= \mathbf{3147 \text{ kg/s}} \end{aligned}$$

(d) The work output from the turbines is

$$\begin{aligned} w_{T,\text{out}} &= h_5 - yh_6 - zh_7 - (1-y-z)h_8 \\ &= 3900 - (0.1446)(3406) - (0.09810)(2918) - (1 - 0.1446 - 0.09810)(2477) \\ &= 1245.4 \text{ kJ/kg} \end{aligned}$$

The net work output from the cycle is

$$\begin{aligned} w_{\text{net}} &= w_{T,\text{out}} - w_{P,\text{in}} \\ &= 1245.4 - 5.1 = 1240.3 \text{ kJ/kg} \end{aligned}$$

The net power output is

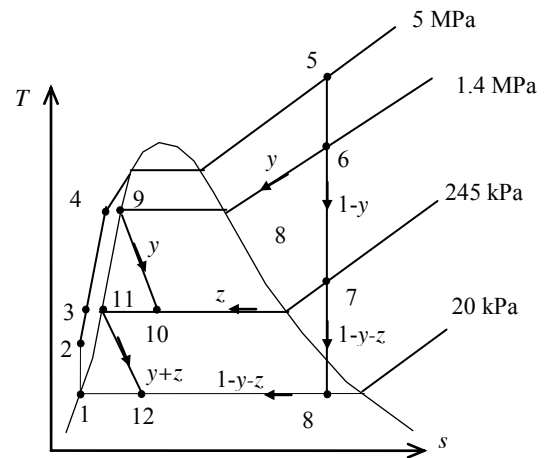
$$\dot{W}_{\text{net}} = \dot{m} w_{\text{net}} = (75 \text{ kg/s})(1240.3 \text{ kJ/kg}) = 93,024 \text{ kW} = \mathbf{93.0 \text{ MW}}$$

The rate of heat input in the boiler is

$$\dot{Q}_{\text{in}} = \dot{m}(h_5 - h_4) = (75 \text{ kg/s})(3900 - 830) \text{ kJ/kg} = 230,250 \text{ kW}$$

The thermal efficiency is then

$$\eta_{\text{th}} = \frac{\dot{W}_{\text{net}}}{\dot{Q}_{\text{in}}} = \frac{93,024 \text{ kW}}{230,250 \text{ kW}} = 0.404 = \mathbf{40.4\%}$$



**10-69** A cogeneration plant is to generate power and process heat. Part of the steam extracted from the turbine at a relatively high pressure is used for process heating. The net power produced and the utilization factor of the plant are to be determined.

**Assumptions** 1 Steady operating conditions exist. 2 Kinetic and potential energy changes are negligible.

**Analysis** From the steam tables (Tables A-4, A-5, and A-6),

$$h_1 = h_f @ 10 \text{ kPa} = 191.81 \text{ kJ/kg}$$

$$v_1 = v_f @ 10 \text{ kPa} = 0.00101 \text{ m}^3/\text{kg}$$

$$\begin{aligned} w_{pI, \text{in}} &= v_1(P_2 - P_1) \\ &= (0.00101 \text{ m}^3/\text{kg})(1200 - 10 \text{ kPa}) \left( \frac{1 \text{ kJ}}{1 \text{ kPa} \cdot \text{m}^3} \right) \\ &= 1.20 \text{ kJ/kg} \end{aligned}$$

$$h_2 = h_1 + w_{pI, \text{in}} = 191.81 + 1.20 = 193.01 \text{ kJ/kg}$$

$$h_3 = h_f @ 1.2 \text{ MPa} = 798.33 \text{ kJ/kg}$$

Mixing chamber:

$$\dot{E}_{\text{in}} - \dot{E}_{\text{out}} = \Delta \dot{E}_{\text{system}} \stackrel{\text{no (steady)}}{=} 0 \longrightarrow \dot{E}_{\text{in}} = \dot{E}_{\text{out}}$$

$$\sum \dot{m}_i h_i = \sum \dot{m}_e h_e \longrightarrow \dot{m}_4 h_4 = \dot{m}_2 h_2 + \dot{m}_3 h_3$$

$$\text{or, } h_4 = \frac{\dot{m}_2 h_2 + \dot{m}_3 h_3}{\dot{m}_4} = \frac{(22.50)(192.41) + (7.50)(798.33)}{30} = 344.34 \text{ kJ/kg}$$

$$v_4 \cong v_f @ h_f = 344.34 \text{ kJ/kg} = 0.001031 \text{ m}^3/\text{kg}$$

$$\begin{aligned} w_{pII, \text{in}} &= v_4(P_5 - P_4) \\ &= (0.001031 \text{ m}^3/\text{kg})(4000 - 1200 \text{ kPa}) \left( \frac{1 \text{ kJ}}{1 \text{ kPa} \cdot \text{m}^3} \right) \\ &= 2.89 \text{ kJ/kg} \end{aligned}$$

$$h_5 = h_4 + w_{pII, \text{in}} = 344.34 + 2.89 = 347.22 \text{ kJ/kg}$$

$$\left. \begin{array}{l} P_6 = 4 \text{ MPa} \\ T_6 = 500^\circ\text{C} \end{array} \right\} \begin{array}{l} h_6 = 3446.0 \text{ kJ/kg} \\ s_6 = 7.0922 \text{ kJ/kg} \cdot \text{K} \end{array}$$

$$\left. \begin{array}{l} P_7 = 1.2 \text{ MPa} \\ s_7 = s_6 \end{array} \right\} h_7 = 3080.4 \text{ kJ/kg}$$

$$\left. \begin{array}{l} P_8 = 10 \text{ kPa} \\ s_8 = s_6 \end{array} \right\} \begin{array}{l} x_8 = \frac{s_8 - s_f}{s_{fg}} = \frac{7.0922 - 0.6492}{7.4996} = 0.8591 \\ h_8 = h_f + x_8 h_{fg} = 191.81 + (0.8591)(2392.1) = 2246.9 \text{ kJ/kg} \end{array}$$

Then,

$$\begin{aligned} \dot{W}_{T, \text{out}} &= \dot{m}_6(h_6 - h_7) + \dot{m}_8(h_7 - h_8) \\ &= (55 \text{ kg/s})(3446.0 - 3080.4) \text{ kJ/kg} + (0.75 \times 55 \text{ kg/s})(3080.4 - 2246.9) \text{ kJ/kg} = 54,494 \text{ kW} \end{aligned}$$

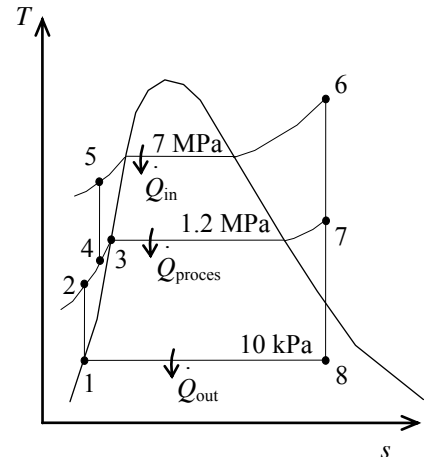
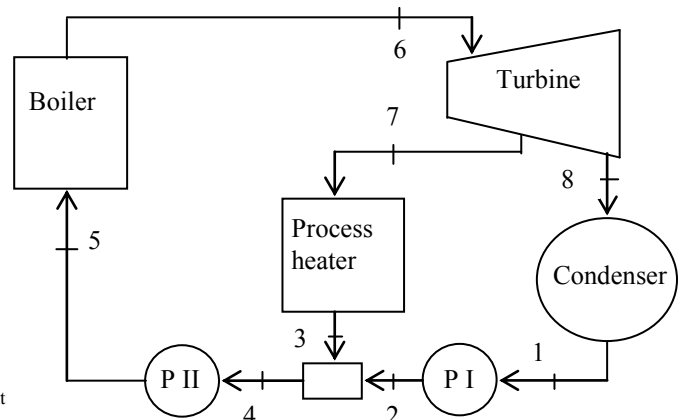
$$\dot{W}_{p, \text{in}} = \dot{m}_1 w_{pI, \text{in}} + \dot{m}_4 w_{pII, \text{in}} = (0.75 \times 55 \text{ kg/s})(1.20 \text{ kJ/kg}) + (55 \text{ kg/s})(2.89 \text{ kJ/kg}) = 208.3 \text{ kW}$$

$$\dot{W}_{\text{net}} = \dot{W}_{T, \text{out}} - \dot{W}_{p, \text{in}} = 54,494 - 208.3 = \mathbf{54,285 \text{ kW}}$$

$$\text{Also, } \dot{Q}_{\text{process}} = \dot{m}_7(h_7 - h_3) = (0.25 \times 55 \text{ kg/s})(3080.4 - 798.33) \text{ kJ/kg} = 31,379 \text{ kW}$$

$$\dot{Q}_{\text{in}} = \dot{m}_5(h_6 - h_5) = (55 \text{ kg/s})(3446.0 - 347.22) = 170,435 \text{ kW}$$

$$\text{and } \varepsilon_u = \frac{\dot{W}_{\text{net}} + \dot{Q}_{\text{process}}}{\dot{Q}_{\text{in}}} = \frac{54,285 + 31,379}{170,435} = 0.5026 = \mathbf{50.3\%}$$



**10-72** A cogeneration plant modified with regeneration is to generate power and process heat. The mass flow rate of steam through the boiler for a net power output of 25 MW is to be determined.

**Assumptions** 1 Steady operating conditions exist. 2 Kinetic and potential energy changes are negligible.

**Analysis** From the steam tables (Tables A-4, A-5, and A-6),

$$h_1 = h_f @ 10 \text{ kPa} = 191.81 \text{ kJ/kg}$$

$$\nu_1 = \nu_f @ 10 \text{ kPa} = 0.00101 \text{ m}^3/\text{kg}$$

$$\begin{aligned} w_{pI, \text{in}} &= \nu_1 (P_2 - P_1) \\ &= (0.00101 \text{ m}^3/\text{kg})(1600 - 10 \text{ kPa}) \left( \frac{1 \text{ kJ}}{1 \text{ kPa} \cdot \text{m}^3} \right) \\ &= 1.61 \text{ kJ/kg} \end{aligned}$$

$$h_2 = h_1 + w_{pI, \text{in}} = 191.81 + 1.61 = 193.41 \text{ kJ/kg}$$

$$h_3 = h_4 = h_9 = h_f @ 1.6 \text{ MPa} = 858.44 \text{ kJ/kg}$$

$$\nu_4 = \nu_f @ 1.6 \text{ MPa} = 0.001159 \text{ m}^3/\text{kg}$$

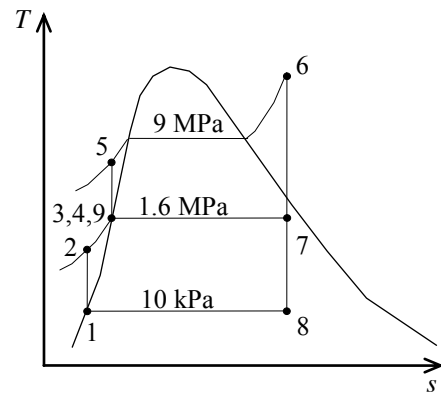
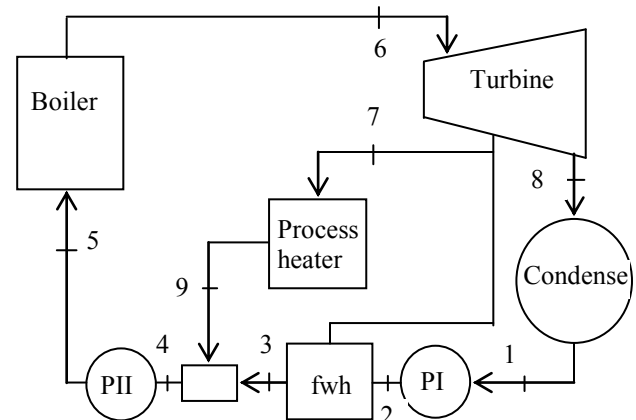
$$\begin{aligned} w_{pII, \text{in}} &= \nu_4 (P_5 - P_4) \\ &= (0.001159 \text{ m}^3/\text{kg})(9000 - 400 \text{ kPa}) \left( \frac{1 \text{ kJ}}{1 \text{ kPa} \cdot \text{m}^3} \right) \\ &= 8.57 \text{ kJ/kg} \end{aligned}$$

$$h_5 = h_4 + w_{pII, \text{in}} = 858.44 + 8.57 = 867.02 \text{ kJ/kg}$$

$$\left. \begin{array}{l} P_6 = 9 \text{ MPa} \\ T_6 = 400^\circ\text{C} \end{array} \right\} \begin{array}{l} h_6 = 3118.8 \text{ kJ/kg} \\ s_6 = 6.2876 \text{ kJ/kg} \cdot \text{K} \end{array}$$

$$\left. \begin{array}{l} P_7 = 1.6 \text{ MPa} \\ s_7 = s_6 \end{array} \right\} \begin{array}{l} x_7 = \frac{s_7 - s_f}{s_{fg}} = \frac{6.2876 - 2.3435}{4.0765} = 0.9675 \\ h_7 = h_f + x_7 h_{fg} = 858.44 + (0.9675)(1934.4) = 2730.0 \text{ kJ/kg} \end{array}$$

$$\left. \begin{array}{l} P_8 = 10 \text{ kPa} \\ s_8 = s_6 \end{array} \right\} \begin{array}{l} x_8 = \frac{s_8 - s_f}{s_{fg}} = \frac{6.2876 - 0.6492}{7.4996} = 0.7518 \\ h_8 = h_f + x_8 h_{fg} = 191.81 + (0.7518)(2392.1) = 1990.2 \text{ kJ/kg} \end{array}$$



Then, per kg of steam flowing through the boiler, we have

$$\begin{aligned} w_{T, \text{out}} &= (h_6 - h_7) + (1 - y)(h_7 - h_8) \\ &= (3118.8 - 2730.0) \text{ kJ/kg} + (1 - 0.35)(2730.0 - 1990.2) \text{ kJ/kg} \\ &= 869.7 \text{ kJ/kg} \end{aligned}$$

$$\begin{aligned} w_{p, \text{in}} &= (1 - y)w_{pI, \text{in}} + w_{pII, \text{in}} \\ &= (1 - 0.35)(1.61 \text{ kJ/kg}) + (8.57 \text{ kJ/kg}) \\ &= 9.62 \text{ kJ/kg} \end{aligned}$$

$$w_{\text{net}} = w_{T, \text{out}} - w_{p, \text{in}} = 869.7 - 9.62 = 860.1 \text{ kJ/kg}$$

Thus,

$$\dot{m} = \frac{\dot{W}_{\text{net}}}{w_{\text{net}}} = \frac{25,000 \text{ kJ/s}}{860.1 \text{ kJ/kg}} = \mathbf{29.1 \text{ kg/s}}$$