

58/60

MET 350

HW #3

ML SUNDERLAND

17

11-58) + 2 stage R-134a

Pressure limits 1.4 MPa and 160 kPa

$$P_3 = 1.4 \text{ MPa}$$

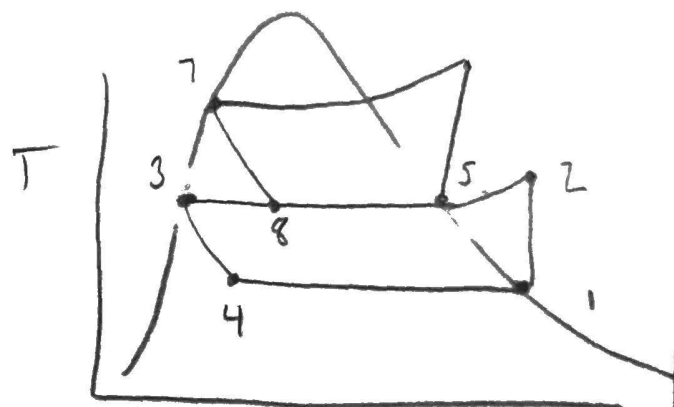
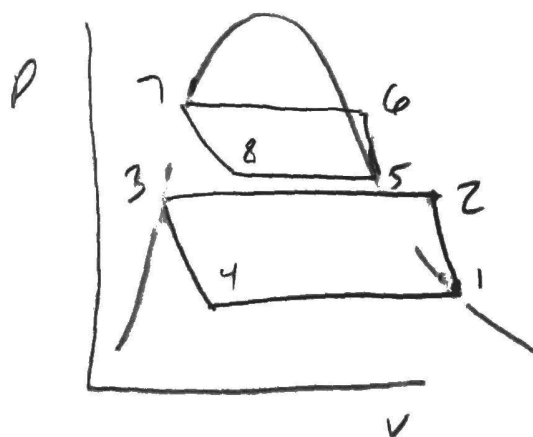
GIVEN: $P_4 = 0.5 \text{ MPa}$

Saturated liquid @ condenser exit
Saturated vapor @ compressor inlet

$$\dot{m}_{r, \text{lower}} = 11 \text{ kg/s}$$

$$R_L = 0.8$$

FIND) a) $\dot{m}_{r, \text{upper}}$; b) $\dot{Q}_L$; c) COP



1 $P_1 = 160 \text{ kPa}$ $x_1 = 1$ $h_1 = 241.14 \text{ kJ/kg}$ $s_1 = 0.94202 \text{ kJ/kg}\cdot\text{K}$ 5

2 $P_2 = 0.5 \text{ MPa}$ $h_{2s} = 259.88 \text{ kJ/kg}$; $h_2 = 264.57 \text{ kJ/kg}$

3 $P_3 = 0.5 \text{ MPa}$ $x_3 = 0$ $h_3 = 63.92 \text{ kJ/kg}$

4 $P_4 = 160 \text{ kPa}$ $h_4 = 63.92 \text{ kJ/kg}$

5 $P_5 = 0.5 \text{ MPa}$ $x_5 = 1$ $h_5 = 259.36 \text{ kJ/kg}$ $s_5 = 0.92420 \text{ kJ/kg}\cdot\text{K}$

6 $P_6 = 1.4 \text{ MPa}$ $h_{6s} = 280.62 \text{ kJ/kg}$ $h_6 = 285.94 \text{ kJ/kg}$

7 $P_7 = 1.4 \text{ MPa}$ $x_7 = 0$ $h_7 = 127.25 \text{ kJ/kg}$

8 $P_8 = 0.5 \text{ MPa}$ $h_8 = 127.25 \text{ kJ/kg}$

$$\eta_c = \frac{w_{in, isentropic}}{w_{in, actual}} = \frac{h_{2s} - h_1}{h_2 - h_1}$$

$$\eta_c (h_2 - h_1) = h_{2s} - h_1$$

$$\eta_c h_2 - \cancel{\eta_c h_1} = h_{2s} - h_1 + \eta_c h_1$$

$$h_2 - \cancel{h_1} = \frac{h_{2s} - h_1}{\eta_c} + h_1 = \frac{259.88 - 241.14}{.8} + 241.14$$

$$h_2 = 264.57 \text{ kJ/kg}$$

$$h_6 = \frac{h_{6s} - h_5}{\eta_c} + h_5 = \frac{280.62 - 259.36}{.8} + 259.36$$

$$h_6 = 285.94 \text{ kJ/kg}$$

$$m_u (h_5 - h_6) = m_L (h_2 - h_3)$$

$$m_u = \frac{.11 (264.57 - 63.92)}{259.36 - 127.25} = \frac{.11 (200.77)}{132.11}$$

$$m_u = .17 \text{ kg/s}$$

$$\dot{Q}_L = m_L (h_1 - h_4) = .11 (241.14 - 63.92)$$

$$\dot{Q}_L = 19.49 \text{ kW}$$

$$\text{COP}_R = \frac{\dot{Q}_L}{\dot{W}_{in, net}} = \frac{\dot{Q}_L}{m_u (h_6 - h_5) + m_L (h_2 - h_1)}$$

$$\text{COP}_R = \frac{19.49}{4.519 + 2.57} = \frac{19.49}{.17 (285.94 - 259.36) + .11 (264.57 - 241.14)}$$

$$= 2.75$$

11-60 | 2 Evap, R-134a

$$T_{EV1} = 0^\circ\text{C}$$

$$T_{EV2} = -26.4^\circ\text{C}$$

$$P_{\text{cond}} = 800 \text{ kPa}$$

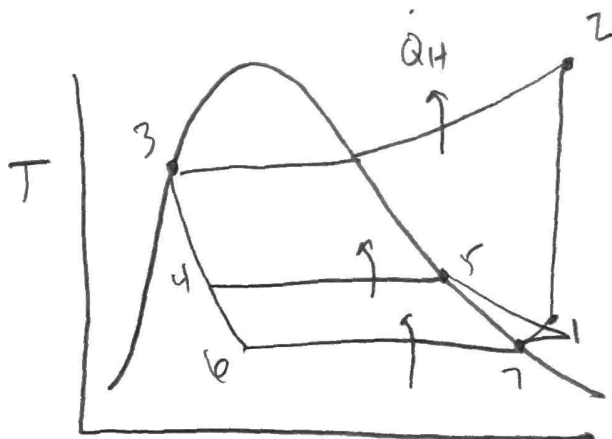
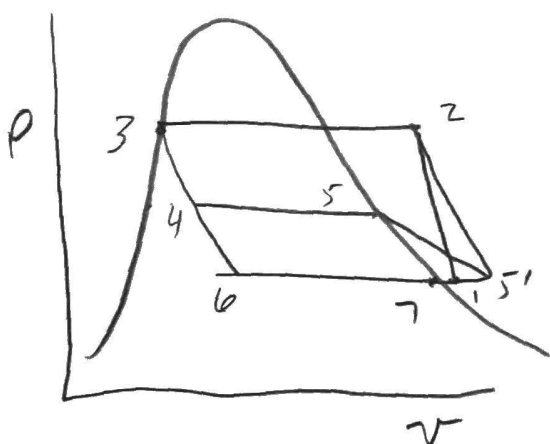
$$\dot{Q}_{L,2} = 8 \text{ kW}$$

$$\dot{m}_R = 0.1 \text{ kg/s}$$

Saturated liquid @ cond exit

Saturated vapor @ exit of evaps

Compressor is isentropic



$$1 \quad s_1 = s_2 \quad h_1 = 241.25 \text{ kJ/kg} \quad T_1 = -26.4^\circ\text{C} \quad P \approx 100 \text{ kPa} \quad s_1 = 0.9788 \text{ kJ/kg}\cdot\text{K}$$

$$2 \quad P_2 = 800 \text{ kPa}; \quad s_2 = s_1, \quad s_2 = 0.9788 \text{ kJ/kg}\cdot\text{K} \Rightarrow h_2 = 286.23 \text{ kJ/kg}$$

$$3 \quad P_3 = 800 \text{ kPa} \quad x_3 = 0; \quad h_3 = 95.48 \text{ kJ/kg} \quad s_3 = 0.35408 \text{ kJ/kg}\cdot\text{K}$$

$$4 \quad T_4 = 0^\circ\text{C} \quad h_4 = 95.48 \text{ kJ/kg}$$

$$5 \quad T_5 = 0^\circ\text{C} \quad x_5 = 1 \quad h_5 = 250.50 \text{ kJ/kg} \quad s_5 = 0.93158 \text{ kJ/kg}\cdot\text{K}$$

$$6 \quad T_6 = -26.4^\circ\text{C} \quad h_6 = 95.48 \text{ kJ/kg}$$

$$7 \quad T_7 = -26.4^\circ\text{C} \quad x_7 = 1; \quad h_7 = 234.44 \text{ kJ/kg} \quad s_7 = 0.95152 \text{ kJ/kg}\cdot\text{K}$$

$$Q_{L,2} = \dot{m}_2 (h_7 - h_6)$$

$$\dot{m}_2 = \frac{Q_L}{h_7 - h_6} = \frac{8 \text{ kW}}{234.44 - 95.48} = 0.057 \text{ kg/s}$$

$$\dot{m}_{\text{ref}} = \dot{m}_1 + \dot{m}_2$$

$$\dot{m}_1 = \dot{m}_{\text{ref}} - \dot{m}_2 = .1 - .057 = 0.043 \text{ kg/s}$$

$$Q_{L,1} = \dot{m}_1 (h_5 - h_4)$$

$$Q_{L,1} = 0.043 (250.5 - 95.48) = \boxed{6.67 \text{ kW}}$$

$$h_1 = \frac{\dot{m}_1 h_5 + \dot{m}_2 h_7}{\dot{m}} = \frac{(0.04243)(250.5) + (.0575)(234.44)}{0.1 \text{ kg/s}}$$

$$h_1 = 241.25 \text{ kJ/kg}$$

$$\dot{W}_{\text{in}} = \dot{m} (h_2 - h_1) = .1 (286.23 - 241.25)$$

$$\boxed{\dot{W}_{\text{in}} = 4.498 \text{ kW}}$$

$$\text{COP}_R = \frac{\dot{Q}_L}{\dot{W}_{\text{in}}} = \frac{8 \text{ kW} + 6.67 \text{ kW}}{4.498 \text{ kW}} = \boxed{3.24}$$